

# Optimization of a novel cryogenic CO<sub>2</sub> capture process by response surface methodology (RSM)

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## **Abstracts**

CO<sub>2</sub> capture and storage (CCS) technologies play a significant role in greenhouse gas (GHG) control. In our previous work, a novel cryogenic CO<sub>2</sub> capture process based on free piston Stirling coolers (FPSCs) was developed. In order to improve capture efficiency, the exploited system was optimized using response surface methodology (RSM). The influence of capture conditions on performance was investigated based on three levels and variables and in central composite design (CCD). The parameters contain flow rate ( $X_1$ : 1 ~ 3 L/min), temperature of FPSC-1 ( $X_2$ : -30 ~ -10 °C) and idle operating time ( $X_3$ : 3 ~ 5 h). The objective of this work is to ascertain the optimal performance of the system (with maximum CO<sub>2</sub> recovery, CO<sub>2</sub> productivity and minimum energy consumption). The experimental data was fitted to a second-order polynomial equation using multiple regression analysis and analyzed using analysis of variance (ANOVA). The dimensional response surface plots and the contour plots derived from the mathematical models were utilized to determine optimum conditions. Results indicate the optimum conditions were: flow rate of 2.16 L/min, temperature of FPSC-1 of -18 °C and operating time of 3.9 h. Under these conditions, the whole process can capture 95.20 % CO<sub>2</sub> with 0.52 MJ/kg captured CO<sub>2</sub> input electricity. Meanwhile, the CO<sub>2</sub> productivity is 44.37 kg CO<sub>2</sub>/h.

**Keywords:** Cryogenic, CO<sub>2</sub> capture, response surface methodology, CO<sub>2</sub> recovery, energy consumption, CO<sub>2</sub> productivity

## **Nomenclature**

*X1* Flow rate of flue gas, L/min

*X2* Temperature of FPSC-1, °C

*X3* Idle operating time, h

$\eta$  CO<sub>2</sub> recovery

$\varphi$  CO<sub>2</sub> productivity

## *Abbreviations*

*ANOVA* Analysis of variance

*CCD* Central composite design

*CCS* CO<sub>2</sub> capture and storage

*CFZ* Controlled freeze zone

*EC* Energy consumption

*FPSC* Free piston Stirling cooler

*GHG* Greenhouse gas

*RSM* Response surface methodology

*TPSA* Temperature pressure swing adsorption

## 1. Introduction

Climate issues have attracted more and more attention in recent decades. It is known that greenhouse gas (GHG) is one of the most important influences on climate change [1]. The major species of GHG include methane ( $\text{CH}_4$ ), carbon dioxide ( $\text{CO}_2$ ), nitrous oxide ( $\text{N}_2\text{O}$ ) and fluorinated gases (HFCs, PFCs and  $\text{SF}_6$ ) [2]. Of these,  $\text{CO}_2$  contributes about 70 % to the enhanced greenhouse effect and global warming, and therefore needs primary mitigation [3]. The emission of  $\text{CO}_2$  is accompanied by the burning of fossil fuel from large fixed industrial stations (*i.e.* coal-fired power plants, steel and cement plants *etc.*). At present, the main post combustion  $\text{CO}_2$  capture technologies include: absorption, adsorption, membrane separation and cryogenic fractionation [4]. Although current commercial technologies such as amine based scrubbing are available for  $\text{CO}_2$  capture from flue gas, the capital and operating cost of capture is still too high [5-8]. This has become a major barrier for the application of  $\text{CO}_2$  capture in power plant sectors and other  $\text{CO}_2$  emitters. Therefore, more research is required to improve  $\text{CO}_2$  capture efficiency and reduce capture costs [9,10].

As an alternative to existing methods, cryogenic separation technologies have also attracted attention in recent decades. Holmes and Ryan (1982) developed a cryogenic distillative process to separate acid gases (mainly  $\text{CO}_2$ ) from methane [11]. Thomas and Denton (1988) proposed a controlled freeze zone (CFZ) process to treat high  $\text{CO}_2/\text{N}_2$  content natural gas, and the process combined two low temperature distillation and a  $\text{CO}_2$  solidification units [12]. Clodic and Younes (2002) built an

anti-sublimation CO<sub>2</sub> capture process named AnSU<sup>®</sup> [13]. In their process, the flue gas was first chilled to the CO<sub>2</sub> freezing point and then the CO<sub>2</sub> in the gas stream frosted on the cold surface of the heat exchanger. Tuinier *et al.* (2011, 2012) exploited a cryogenic packed bed to recover CO<sub>2</sub> from the flue gas and purify the biogas [14,15]. During the process, H<sub>2</sub>O and CO<sub>2</sub> can be separated from the different locations in the bed by the difference in freezing point. Berstad *et al.* (2012) put forward a low temperature distillation process to remove CO<sub>2</sub> from natural gas [16]. The CO<sub>2</sub> concentration of the natural gas was reduced from an initial 50.6% to 50 ppm by three distillation columns, and the purity of the final CO<sub>2</sub> product is 94.35%. It should be noted that the intricate phase variation (liquefaction or solidification) and thermodynamic process (mass and heat transfer) usually accompany with the designed cryogen capture processes [17]. In order to improve capture efficiency and minimize energy consumption, a good understanding of the complex relationships among the operation parameters involved in the cryogenic CO<sub>2</sub> capture processes is necessary.

In light of these concerns, an effective approach that reveals the effect of key process parameters and their interactions on CO<sub>2</sub> capture performance is particularly significant. Response surface methodology (RSM) is a statistical technique for optimizing complex processes due to its more efficient and easier arrangement of experiments [18]. This method is less laborious and time-consuming than other approaches applied to optimize a process. Although a reduced number of experimental trials are needed to evaluate multiple factors and their interactions, they are helpful to determine the target value. Hence, RSM provides an effective tool for

investigating aspects that affect the desired response if there are many factors and interactions in the experiment. To optimize the process, RSM can be employed to determine a suitable polynomial equation for describing the response surface. Several published studies have investigated the potential application of RSM to CO<sub>2</sub> capture processes. Serna-Guerrero *et al.* (2010) determined the influence of desorption pressure, desorption temperature, gas flow rate, and their corresponding interactions on the regeneration performance (working capacity and desorption rate) of an amine-based CO<sub>2</sub> adsorption process by RSM [19]. Mulgundmath and Tezel (2010) investigated the influence of four control parameters (*i.e.* purge/feed flow ratio, purge time, purge gas temperature and adsorption pressure) on CO<sub>2</sub> recovery in a temperature pressure swing adsorption (TPSA) system [20]. Nuchitprasittichai and Cremasch (2011) optimized the amine based CO<sub>2</sub> capture process by RSM [21]. The impacts of the absorber and stripper column heights, the concentration of amine solvents, and operating condition of the CO<sub>2</sub> recovery and energy consumption with various amine solvents was studied in detail. García *et al.* (2011) used the RSM method to investigate the influence of the adsorption CO<sub>2</sub> partial pressure and temperature on CO<sub>2</sub> capture capacity and the breakthrough time of activated carbon [22]. The combined effects of the CO<sub>2</sub> partial pressure and temperature on CO<sub>2</sub> capture capacity and breakthrough time of the adsorption process was evaluated.

The objective of this work is to optimize the cryogenic CO<sub>2</sub> capture efficiency of the free piston Stirling cooler (FPSC) system using RSM. The vital parameters that affect the capture performance have been identified as the flow rate of flue gas,

temperature of FPSC-1 (which is used to chill the flue gas and separate H<sub>2</sub>O) and idle operating time before gas inflow [23]. On the other hand, CO<sub>2</sub> recovery, energy consumption and CO<sub>2</sub> productivity are representative of system performance.

The paper is organized as follows: Section 2 introduces the CO<sub>2</sub> capture process based on the FPSCs system. Section 3 describes the structure of the system and experiment design by RSM. Section 4 discusses the impact of parameters on CO<sub>2</sub> recovery, energy consumption and CO<sub>2</sub> productivity. Section 5 summarizes the optimum condition of the novel cryogenic CO<sub>2</sub> capture process.

## **2. Cryogenic CO<sub>2</sub> capture process based on FPSCs**

The schematic of the cryogenic process is shown in Fig. 1. The whole process can be divided into 3 sections: 1) cryogenic unit 1 (C-1); 2) cryogenic unit 2 (C-2); 3) cryogenic unit 3 (C-3) [24].

### *2.1. Cryogenic unit 1 (C-1)*

First, flue gas is chilled by FPSC-1 in the cryogenic unit 1 (C-1). At C-1, the moisture in the feed gas condenses into water and then flows out from the outlet to avoid clogging the vessel. This is the key issue of cryogenic CO<sub>2</sub> separation technologies. The dry flue gas is prechilled to a low temperature to facilitate CO<sub>2</sub> anti-sublimation in the subsequent stage. In order to improve the exergy efficiency of the whole process, the latent and sensible heat of the condensate water is recuperated by the heat exchanger with the incoming hot flue gas.

## 2.2. Cryogenic unit 2 (C-2)

In cryogenic unit 2 (C-2), FPSC-2 provides the cryogenic condition, and the flue gas is cooled down to below  $-100^{\circ}\text{C}$ . According to the work of Clodic *et al.*, the freezing point of  $\text{CO}_2$  for the flue gas from a typical coal-fired power plant (typically 3 ~ 20 %  $\text{CO}_2$ ) varies in the range of  $-112 \sim -97^{\circ}\text{C}$  [13]. In our research, the percentage of  $\text{CO}_2$  in flue gases is about 13 vol%, and which consequently has a frost point of approximately  $-100^{\circ}\text{C}$ . When the gas stream passes through the low temperature cold head, the  $\text{CO}_2$  immediately solidifies into dry ice and frosts on the surface of the cold head. In comparison, other gas (such as  $\text{N}_2$ ) is exhausted without phase change. Meanwhile, the sensible heat of the cold residual gas is also recovered by the subsequent flue gas.

## 2.3. Cryogenic unit 3 (C-3)

In this section, a motor driven scraping rod is utilized to separate the deposited  $\text{CO}_2$  from the surface of the cold head, and the captured  $\text{CO}_2$  is gathered in cryogenic unit 3 (C-3), where FPSC-3 provides a low temperature condition (below  $-78.5^{\circ}\text{C}$ ) to store dry ice and prevent it gasifying. In order to separate  $\text{CO}_2$  from the residual gas, the frosted  $\text{CO}_2$  is temporarily stored in C-3. Since the latent heat of the frosted  $\text{CO}_2$  is substantial, its cold energy is recovered by the heat exchanger. During the  $\text{CO}_2$  sublimation process, an amount of sensible heat is required and is absorbed from the hot gas stream. Thus, the incoming flue gas is sufficiently chilled before pumping into the cryogenic units.



### **3. Experimental**

#### *3.1. Apparatus*

The structure of the experimental apparatus is shown in Fig. 2. The system detail has been described in previous work [25,26]. It is noteworthy that a couple of improvements in the system have been carried out. First, the tower body and its junctions have been wrapped with thermal insulation material (expand aple poly ephylene); second, the FPSC-1 was replaced by a low power unit (80 W) to save input electricity.

The composition of the flue gas from a typical coal-fired power plant is simulated by CO<sub>2</sub>/13%, H<sub>2</sub>O/5% and N<sub>2</sub>/82%. The detailed condition of the gas mixture can be found in [25].

#### *3.2. Experimental design*

The capture performance based on FPSCs system was optimized by RSM packages in Design Expert 7.0.10. As a convenient statistical tool, Design Expert offers multilevel factorial screening designs, and numerical optimization can be realized by analyzing the critical factors and their interactions. The design of runs was in accordance with central composite design (CCD). Based on single factor experimental results, the three major influence factors were clarified as parameters of flow rate, temperature of FPSC-1 and idle operating time before gas inflow, and each factor was manipulated in the range of 1 ~ 3 L/min, -30 ~ -10 °C and 3 ~ 5 h, respectively.

In addition, the energy consumption (EC) of the cryogenic system per unit mass CO<sub>2</sub> captured is defined as follows [27]:

$$\text{Energy consumption (EC)} = \frac{UI}{\left(\frac{v_{in} \omega_{CO_2,in} P_{in}}{T_{in}} - \frac{v_{out} \omega_{CO_2,out} P_{out}}{T_{out}}\right)} \cdot \frac{nR}{M_{CO_2}} \quad (1)$$

in which  $U$  and  $I$  are voltage and current in the system, respectively.  $v_{CO_2, in}$  and  $v_{CO_2, out}$  are volume flow rates of the gas mixture at the inlet and outlet.  $P$  and  $T$  are pressure and temperature at the inlet and outlet.  $M_{CO_2}$  is the molar mass of CO<sub>2</sub>.

The CO<sub>2</sub> capture efficiency is defined as:

$$\text{CO}_2 \text{ recovery } (\eta) = 1 - \frac{v_{out} \omega_{CO_2,out} P_{out} T_{in}}{v_{in} \omega_{CO_2,in} P_{in} T_{out}} \quad (2)$$

where  $\eta$  is CO<sub>2</sub> capture efficiency;  $v$  is flow rate of gas mixture and  $\omega$  is percentage of CO<sub>2</sub> in gas mixture.

Meanwhile, the CO<sub>2</sub> productivity can be calculated as:

$$\text{CO}_2 \text{ productivity } (\varphi) = (v_{in} \cdot \omega_{CO_2,in} - v_{out} \cdot \omega_{CO_2,out}) \cdot \rho_{CO_2} \quad (3)$$

where  $\varphi$  is CO<sub>2</sub> productivity;  $\rho$  is density of CO<sub>2</sub>.

The experimental conditions of CCD runs of Design Expert are presented in Table 1. As shown in Table 1, the three factors chosen for this study were designated as  $X_1$ ,  $X_2$  and  $X_3$  categorized into three levels, coded +1, 0, -1 for high, intermediate and low value, respectively. The coded values of the process parameters were determined by the following equation:

$$X_i = \frac{x_i - x_0}{\Delta x}, \quad i = 1, 2, 3 \quad (4)$$

where  $X_i$  is the coded value,  $x_i$  is the corresponding actual value,  $x_0$  is the actual value of the independent variable at the center point, and  $\Delta x$  is the step change of the variable.

The behavior of the system is explained by the following quadratic polynomial equation:

$$Y_k = \beta_0 + \sum_{i=1}^3 \beta_i X_i + \sum_{i=1}^3 \beta_{ki} X_i^2 + \sum_{i=0}^2 \sum_{j=i+1}^3 \beta_{ij} X_i X_j, \quad k = 1, 2; j = 2, 3 \quad (5)$$

where  $Y_k$  is the  $k$  th response function.  $\beta_0$ ,  $\beta_i$ ,  $\beta_{ii}$  and  $\beta_{ij}$  are the coefficients of intercept, linear, quadratic and interactive terms, respectively.  $X_i$  and  $X_j$  represent the  $i$  th and  $j$  th coded independent variables.

## 4. Results and discussion

### 4.1. Statistical analysis

The analysis of variance program (ANOVA) in Design Expert software was used for regression analysis for the obtained data to estimate the coefficient of the regression equation. The fitted polynomial equation was expressed as 3D surface and contour plots in order to visualize the relationship between the responses and experimental levels of each factor and to deduce the optimum conditions. According to the analysis of variance, the effect and regression coefficients of individual linear, quadratic and interaction terms were determined. The regression coefficients were then used to generate dimensional and contour maps from the regression models.

A total of 17 runs for optimizing the three individual parameters in the CCD were undertaken and experimental conditions according to the factorial design are shown in Table 2. Results show that the CO<sub>2</sub> recovery, energy consumption and CO<sub>2</sub> productivity varied in the range of 70.49 to 95.20 %, 0.52 to 2.13 MJ/kg<sub>captured</sub> CO<sub>2</sub> and 15.89 to 45.73 kg CO<sub>2</sub>/h, respectively.

#### *4.2. Results of ANOVA analysis*

##### *4.2.1. CO<sub>2</sub> recovery*

ANOVA was undertaken to obtain the process factors and response. The statistical significance was evaluated using the F-value and P-value, and the lack-of-fit value of the model indicates non-significance as desired. The goodness of fit of the polynomial model was expressed by the determination coefficient R<sup>2</sup>, adjusted R<sup>2</sup> (R<sup>2</sup><sub>adj</sub>), and predicted R<sup>2</sup>. From the ANOVA results of CO<sub>2</sub> recovery (in Table 3), the results indicate the model is significant (P-value < 0.05). The lack-of-fit (4.97) implies that the result is not significant relative to pure error. The measures of R<sup>2</sup>, adjusted R<sup>2</sup> (R<sup>2</sup><sub>adj</sub>), and predicted R<sup>2</sup> are close to 1, which implies an adequate model. The value of the precision ratio, 23.409 indicates adequate model discrimination.

By applying multiple regression analysis to the experimental data, the predicted model of CO<sub>2</sub> recovery was obtained by the following second-order polynomial functions:

$$\begin{aligned}
CO_2 \text{ recovery (\%)} = & 78.29317 - 2.05444 * X_1 + 0.094756 * X_2 + 2.56601 * X_3 \\
& - 0.036137 * X_1 X_2 - 1.04564 * X_1 X_3 + 0.030693 * X_2 X_3 \\
& + 0.75978 * X_1^2 + 4.59801E-003 * X_2^2 + 0.090564 * X_3^2
\end{aligned} \tag{6}$$

#### 4.2.2. Energy consumption

The ANOVA results of energy consumption are listed in Table 4. The model results indicate the model is significant (P-value < 0.05). The lack-of-fit of 0.29 implies that the result is not significant relative to pure error. The measures of  $R^2$ , adjusted  $R^2$  ( $R^2_{adj}$ ), and predicted  $R^2$  are close to 1, which implies an adequate model. The value of the precision ratio, 22.423 indicates adequate model discrimination.

From the discussion above, it is clear that the model of energy consumption has an adequate precision. The obtained quadratic approximating model for energy consumption is described as follows:

$$\begin{aligned}
\text{Energy consumption (MJ/kg}_{\text{captured}} \text{ CO}_2) = & 2.36438 - 2.06351 * X_1 - 0.041158 * X_2 + 0.24684 * X_3 \\
& - 1.82814E-003 * X_1 X_2 - 0.021632 * X_1 X_3 + 3.51065E-003 * X_2 X_3 \\
& + 0.37480 * X_1^2 - 6.89216E-004 * X_2^2 + 2.14984E-003 * X_3^2
\end{aligned} \tag{7}$$

#### 4.2.3. CO<sub>2</sub> productivity

The results of ANOVA analysis for CO<sub>2</sub> productivity are summarized in Table 5. The results indicate the model is significant (P-value < 0.05). The lack-of-fit of 3.85 implies that the result is not significant relative to pure error. The measures of  $R^2$ , adjusted  $R^2$  ( $R^2_{adj}$ ), and predicted  $R^2$  are close to 1, which implies an adequate model.

The value of precision ratio 20.104 indicates adequate model discrimination.

In light of the ANOVA analysis, the predicted model of CO<sub>2</sub> productivity can be calculated by the following second-order polynomial equation:

$$\begin{aligned} CO_2 \text{ productivity (kg / h)} = & 21.43727 + 5.18471 * X_1 + 13.91162 * X_2 - 3.17533 * X_3 \\ & - 1.86241 * X_1 X_2 + 7.74237 * X_1 X_3 + 0.382177 * X_2 X_3 \\ & + 3.74721 * X_1^2 + 0.015271 * X_2^2 + 2.113741 * X_3^2 \end{aligned} \quad (8)$$

#### 4.3. Model validation

In order to confirm the RSM validity, the model equation for predicting the optimum response values was tested using the selected conditions. Three confirmation experiments were implemented with process parameters chosen randomly from the ranges of Table 1 in order to validate the mathematical models. The actual results in terms of the average of three measured results were calculated. Table 6 shows the actual values, predicted values and calculated error of confirmation experiments. Fig. 3 shows respective plots of actual and predicted value of CO<sub>2</sub> recovery (a), energy consumption (b) and CO<sub>2</sub> productivity (c). The results illustrate that the developed models can effectively predict the capture performance (CO<sub>2</sub> recovery, energy consumption and CO<sub>2</sub> productivity) of the cryogenic system.

#### 4.4. Optimization of CO<sub>2</sub> recovery

In this section, process optimization was implemented to find the conditions under which maximum CO<sub>2</sub> recovery is possible. 3D response surface and 2D contour plots indicate the effects of parameters and their interactions on CO<sub>2</sub> recovery. The optimal values of the selected variables were obtained by solving the regression equations.

They show the type of interactions between two tested variables and the relationship between responses and experiment levels of each variable. Two variables within the experimental range are depicted in the 3D surface plots when the third variable was kept constant at zero.

Fig. 4-6 show the results of CO<sub>2</sub> recovery affected by flow rate (X<sub>1</sub>), temperature of FPSC-1 (X<sub>2</sub>) and idle operating time (X<sub>3</sub>). Fig. 4 shows the 3D surface plot and the contour plot of the effect of the flow rate (X<sub>1</sub>) and temperature of FPSC-1 (X<sub>2</sub>) on CO<sub>2</sub> recovery. It is clear that the CO<sub>2</sub> recovery increased from 74.37 to 95.23 % with an increased feed gas of 1.0 to 2.2 L/min. Then, CO<sub>2</sub> recovery decreases from 95.23 to 74.75 % with the flow rate varying from 2.2 to 3.0 L/min. This is because when the flow rate is lower than 2.2 L/min, the cold head can capture the majority of the CO<sub>2</sub> in the gas mixture. However, when the flow rate increased to higher than 2.2 L/min, the amount of CO<sub>2</sub> passed through the low temperature surface of the cold head without anti-sublimation process, and thus the CO<sub>2</sub> recovery decreased dramatically. When the temperature of FPSC-1 dropped from -10 to -18 °C, the CO<sub>2</sub> recovery increased from 75.57 to 95.21%. If the temperature continually decreased to -30 °C, the CO<sub>2</sub> recovery decreased gradually (from 95.21 to 72.11 %).

Fig. 5 shows the effect of the flow rate (X<sub>1</sub>) and idle operating time (X<sub>3</sub>) on CO<sub>2</sub> recovery. It indicates that while the idle operating time decreased from 5.0 to 3.9 h, the CO<sub>2</sub> recovery of the system increased from 73.45 to 95.16 %. In contrast, the idle operating time varied from 3.9 to 3.0 h, the CO<sub>2</sub> recovery decreased to 74.33 %. This

is because when the idle operating time is near 5 h, the system was excessively pre-chilled to solidify H<sub>2</sub>O into ice indirectly, and the ice clogged the vessel. This would adversely affect the anti-sublimation of CO<sub>2</sub>, and then lead to low CO<sub>2</sub> recovery. In comparison, when the idle operating time is too short (around 3 h), a fraction of H<sub>2</sub>O in the flue gas could not be separated effectively and was pumped into the cryogenic unit 2 along with the gas stream. Under the low temperature condition, the H<sub>2</sub>O solidified and frosted on the surface of the cold head, which prevents heat and mass transfer in the CO<sub>2</sub> anti-sublimation stage. From Fig. 5, the effect of flow rate on CO<sub>2</sub> recovery can also be observed. The optimal flow rate of the gas stream is around 2.1 L/min with the a CO<sub>2</sub> recovery of 95.16%, and this is in accordance with the result in Fig. 4.

Fig. 6 shows the effect of the temperature of FPSC-1 ( $X_2$ ) and idle operating time ( $X_3$ ) on CO<sub>2</sub> recovery. The contour plot indicates that when the temperature of FPSC-1 dropped from -10 to -17°C, CO<sub>2</sub> recovery increased gradually (from 79.45 to 95.08%). However, while the temperature steadily decreased to -30°C, CO<sub>2</sub> recovery reduced rapidly. This is because the function of FPSC-1 is to chill the cryogenic unit 1 (C-1) and condense moisture from the flue gas. When the temperature of FPSC-1 is higher than -17 °C, the moisture in the gas stream flowed into C-2 and solidified into ice on the surface of the cold head and adversely affected the anti-sublimation of the incoming CO<sub>2</sub> gas. While the temperature is lower than -17 °C, the flue gas cannot be effectively pre-chilled, and the CO<sub>2</sub> recovery is also reduced. In addition, the influence of the idle operating time on the CO<sub>2</sub> recovery is also depicted in Fig. 6. The



optimal idle operating time is around 4.2 h with CO<sub>2</sub> recovery of 94.87%.

#### 4.5. Optimization of energy consumption

Response surfaces were plotted to study the effects of parameters and their interactions on energy consumption. The results of energy consumption affected by flow rate ( $X_1$ ), temperature of FPSC-1 ( $X_2$ ) and operating time ( $X_3$ ) are shown in Fig. 7-9. These types of plots show effects of two factors on the response when the other factors were kept at zero.

Fig. 7 shows the effect of the flow rate ( $X_1$ ) and temperature of FPSC-1 ( $X_2$ ) on energy consumption. It indicates that the energy consumption decreased rapidly (from 2.41 to 0.53 MJ/kg<sub>captured</sub> CO<sub>2</sub>) when the flue gas flow varied from 1.0 to 1.9 L/min. With increasing flow rate from 1.9 to 3.0 L/min, the energy consumption increased again from 0.53 to 2.86 MJ/kg<sub>captured</sub> CO<sub>2</sub>. This is because when the flow rate is lower than 1.9 L/min, most of the CO<sub>2</sub> in the gas stream can be captured while it passes through the cooling fin of the cold head. However, when the flow rate is too fast (higher than 1.9 L/min), amount of CO<sub>2</sub> cannot be captured effectively and thus energy consumption increased. In addition, the energy consumption basically reduced (from 2.41 to 0.54 MJ/kg<sub>captured</sub> CO<sub>2</sub>) when the temperature of FPSC-1 decreased from -10 to -18 °C. However, when the temperature continuously dropped to -30 °C, the energy consumption dramatically increased to 2.62 MJ/kg<sub>captured</sub> CO<sub>2</sub>.

Fig. 8 shows the effect of flow rate ( $X_1$ ) and idle operating time ( $X_3$ ) on energy consumption. When the flow rate increased from 1.0 to 2.2 L/min, the energy

consumption of the system reduced gradually (from 2.25 to 0.53 MJ/kg<sub>captured CO<sub>2</sub></sub>). However, with the subsequent increase in the flow rate (from 2.2 to 3.0 L/min), the energy consumption increased from 0.53 to 1.94 MJ/kg<sub>captured CO<sub>2</sub></sub>. This is consistent with the obtained optimal flow rate shown in Fig. 7. In addition, the energy consumption can be obviously minimized (from 2.25 to 0.52 MJ/kg<sub>captured CO<sub>2</sub></sub>) when the idle operating time is shortened from 5.0 to 3.8 h. When the idle operating time continually dropped to 3.0 h, the energy consumption dramatically increased to 1.97 MJ/kg<sub>captured CO<sub>2</sub></sub>. This is because when the idle operating time decreased from 5.0 to 3.8 h, the operating cost of the system was significantly reduced while the CO<sub>2</sub> recovery of the system can be still kept at approximately 95 %. In contrast, when the idle operating time is too short (up to 3.0 h), the low temperature of the system cannot reach the condensing and freezing point of the corresponding components (H<sub>2</sub>O and CO<sub>2</sub>), and thus the energy consumption increase.

Fig. 9 illustrates the interaction of the temperature of FPSC-1 ( $X_2$ ) and operating time ( $X_3$ ) on energy consumption. It can be seen that the energy consumption of the system decreased gradually from 1.78 to 0.52 MJ/kg<sub>captured CO<sub>2</sub></sub> with the temperature of FPSC-1 varied from -30 to -20 °C. When the temperature continuously rose from -20 to -10 °C, the energy consumption increased from 0.52 to 1.96 MJ/kg<sub>captured CO<sub>2</sub></sub>. In contrast, the energy consumption decreased from 1.81 to 0.52 MJ/kg<sub>captured CO<sub>2</sub></sub> with idle operating time increasing from 3.0 h to 3.8 h. When the idle operating time continuously extended from 3.8 h to 5.0 h, energy consumption increased from 0.52 to 2.1 MJ/kg<sub>captured CO<sub>2</sub></sub>. Therefore, it can be concluded that the optimal idle operating

time and temperature of FPSC-1 are 3.8 h and -20 °C.

#### *4.6. Optimization of CO<sub>2</sub> productivity*

Fig. 10 shows the 3D surface plot and the contour plot of the effect of the flow rate ( $X_1$ ) and temperature of FPSC-1 ( $X_2$ ) on CO<sub>2</sub> productivity. It is clear that the CO<sub>2</sub> productivity increased from 15.37 to 44.37 kg/h with increasing flow rate of feed gas from 1.0 to 2.0 L/min. Then, the CO<sub>2</sub> productivity decreases from 45.37 to 16.82 kg/h with the flow rate varying from 2.0 to 3.0 L/min. When the flow rate is lower than 2.0 L/min, the majority of the CO<sub>2</sub> in the gas mixture can be captured by the cold head. However, when the flow rate increased to higher than 2.0 L/min, the amount of CO<sub>2</sub> passed through the low temperature surface of the cold head without anti-sublimation process, and thus the CO<sub>2</sub> productivity decreased dramatically. When the temperature of FPSC-1 dropped from -10 to -21 °C, the CO<sub>2</sub> productivity of the process increased from 16.01 to 44.53 kg/h. Therefore, if the temperature continually decreased to -30 °C, the CO<sub>2</sub> productivity decreased gradually (from 44.53 to 16.84 kg/h).

Fig. 11 shows the effect of the flow rate ( $X_1$ ) and idle operating time ( $X_3$ ) on CO<sub>2</sub> productivity. It indicates that while the idle operating time decreased from 5.0 to 4.2 h, the CO<sub>2</sub> productivity of the system increased from 15.79 to 45.21 kg/h. In contrast, when the idle operating time varied from 4.2 to 3.0 h, the CO<sub>2</sub> productivity decreased to 17.12 kg/h. The influence of idle operating time indicates that when the idle operating time is near 5 h, the system was excessively pre-chilled to solidify H<sub>2</sub>O into ice indirectly, and ice clogged the vessel. This would adversely affect the

anti-sublimation of CO<sub>2</sub>, and lead to low CO<sub>2</sub> productivity. In comparison, when the idle operating time is too short (around 3 h), a fraction of H<sub>2</sub>O in the flue gas could not be separated effectively and was pumped into the cryogenic unit 2 along with the gas stream. Under low temperature conditions, the H<sub>2</sub>O solidified and frosted on the surface of the cold head which prevents heat and mass transfer in the CO<sub>2</sub> anti-sublimation stage. In addition, the effect of flow rate on the CO<sub>2</sub> productivity can be observed. It was noted that the optimal flow rate of the gas stream is around 2.2 L/min with the CO<sub>2</sub> productivity of 44.97 kg/h.

Fig. 12 shows the effect of the temperature of FPSC-1 ( $X_2$ ) and idle operating time ( $X_3$ ) on CO<sub>2</sub> productivity. The contour plot indicates that when the temperature of FPSC-1 dropped from -10 to -19 °C, CO<sub>2</sub> productivity increased gradually (from 16.22 to 44.17 kg/h). However, when the temperature continually decreased to -30 °C, CO<sub>2</sub> productivity reduced rapidly. This is because the function of FPSC-1 is to chill the cryogenic unit 1 (C-1) and condense moisture from the flue gas. When the temperature of FPSC-1 is higher than -19 °C, the moisture in the gas stream flowed into C-2 and solidified into ice on the surface of the cold head and adversely affected the anti-sublimation of the incoming CO<sub>2</sub> gas. When the temperature is lower than -19 °C, the flue gas cannot be effectively pre-chilled, and CO<sub>2</sub> productivity also reduces. In addition, the influence of the idle operating time on the CO<sub>2</sub> productivity is depicted in Fig. 12. The optimal idle operating time is around 4.0 h with CO<sub>2</sub> productivity of 44.34 kg/h.

#### *4.7. Prospective for scaling up of the system*

In order to commercially apply the developed system for the industrial emissions, investigation of the feasibility of scaling up the developed cryogenic CO<sub>2</sub> capture process is significant. To process the enormous flow of flue gas in a scaled up plant, the following measures should be carried out: 1) the diameter of separation tower must be correspondingly increased. 2) The number of Stirling coolers in different stages (pre-freezing, main freezing and storage) should also be increased. 3) From the current situation, heat loss would be an unavoidable challenge due to the huge temperature difference between the internal and external features of the installation and can be overcome by an effective heat recovery process.

### **5. Conclusion**

The optimization of the cryogenic CO<sub>2</sub> capture process based on the FPSCs system was analyzed focusing on CO<sub>2</sub> recovery and energy consumption. The optimization employed an RSM experimental design to obtain operational conditions for maximum CO<sub>2</sub> recovery and minimum energy consumption. The optimization results show that the flow rate of feed gas ( $X_1$ ), temperature of FPSC-1 ( $X_2$ ) and idle operating time ( $X_3$ ) of the system have a significant effect on CO<sub>2</sub> recovery and energy consumption. The optimal condition of the system is as follows: flow rate of flue gas is 2.16 L/min, temperature of FPSC-1 is -18 °C and idle operating time is 3.9 h. Under these conditions, 95.20 % CO<sub>2</sub> can be removed from flue gas and energy consumption is 0.52 MJ/kg<sub>captured</sub> CO<sub>2</sub>. Meanwhile, the CO<sub>2</sub> productivity is 44.37 kg CO<sub>2</sub>/h.

It is worth noting that for the RSM approach, the stated value of the independent variables must fall within the prescribed limits. When the value of each factor is beyond the boundary, the developed model will become invalid. In future, a more effective approach (such as a neural network method) should be considered to avoid this disadvantage.

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## Figure captions

Fig. 1 Schematic of cryogenic capture process

Fig. 2 The structure of the exploited cryogenic CO<sub>2</sub> capture system

Fig. 3 Relationship between experimental and predicted values. (a) CO<sub>2</sub> recovery (%), (b) energy consumption (MJ/kg<sub>captured</sub> CO<sub>2</sub>) and (c) CO<sub>2</sub> productivity (kg/h)

Fig. 4 Interaction effect of flow rate ( $X_1$ ) and temperature of FPSC-1 ( $X_2$ ) on CO<sub>2</sub> recovery (%). (a) 3D surface plot; (b) contour plot.

Fig. 5 Interaction effect of flow rate ( $X_1$ ) and operating time ( $X_3$ ) on CO<sub>2</sub> recovery (%). (a) 3D surface plot; (b) contour plot.

Fig. 6 Interaction effect of temperature of FPSC-1 ( $X_2$ ) and operating time ( $X_3$ ) on CO<sub>2</sub> recovery (%). (a) 3D surface plot; (b) contour plot.

Fig. 7 Interaction effect of flow rate ( $X_1$ ) and temperature of FPSC-1 ( $X_2$ ) on energy consumption (MJ/kg<sub>captured</sub> CO<sub>2</sub>). (a) 3D surface plot; (b) contour plot.

Fig. 8 Interaction effect of flow rate ( $X_1$ ) and operating time ( $X_3$ ) on energy consumption (MJ/kg<sub>captured</sub> CO<sub>2</sub>). (a) 3D surface plot; (b) contour plot.

Fig. 9 Interaction effect of temperature of FPSC-1 ( $X_2$ ) and operating time ( $X_3$ ) on energy consumption (MJ/kg<sub>captured</sub> CO<sub>2</sub>). (a) 3D surface plot; (b) contour plot.

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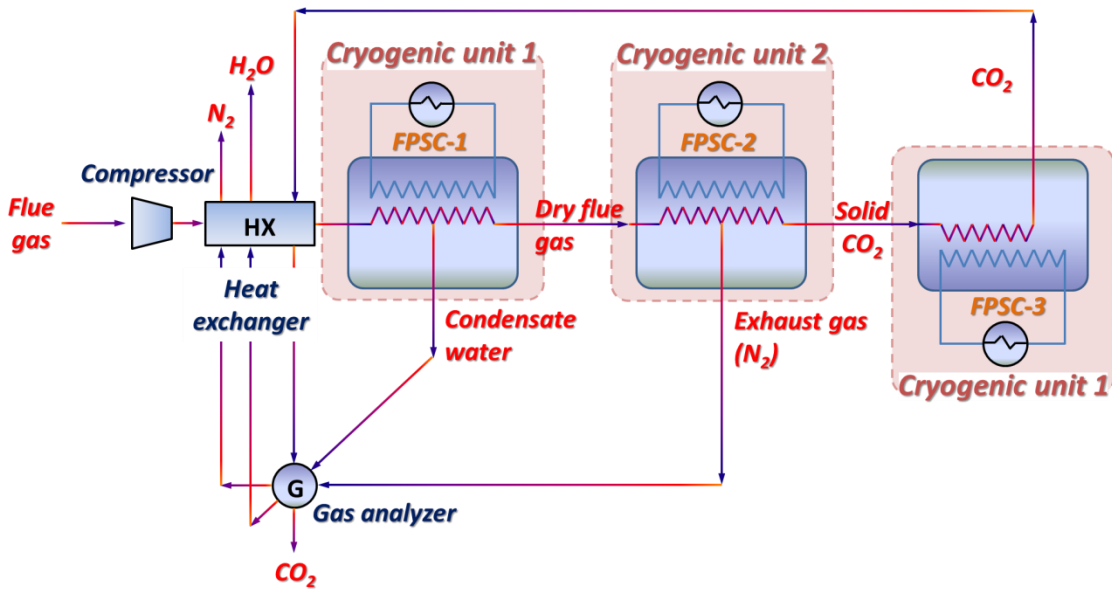


Fig. 1 Schematic of cryogenic capture process

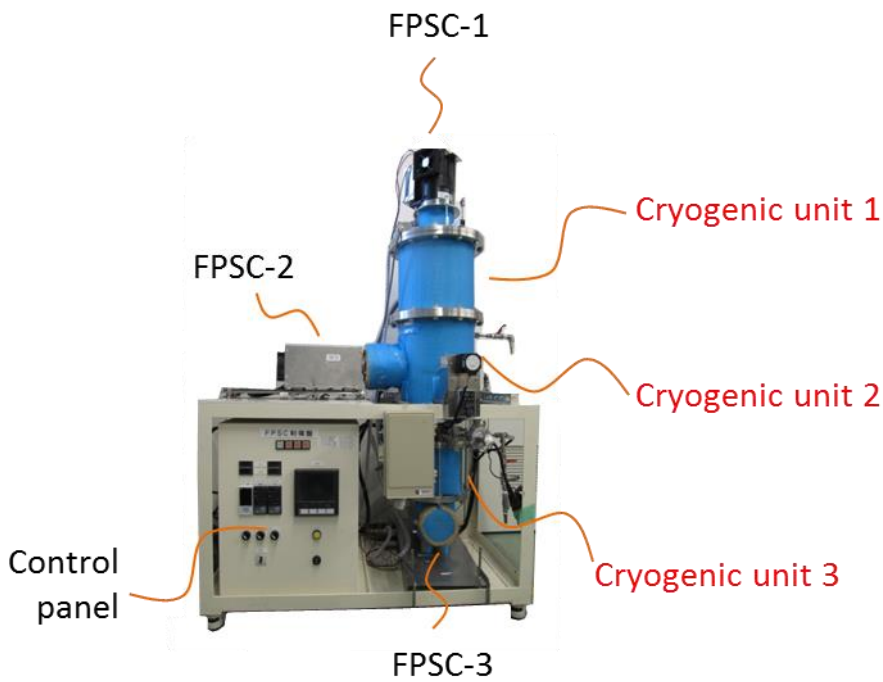
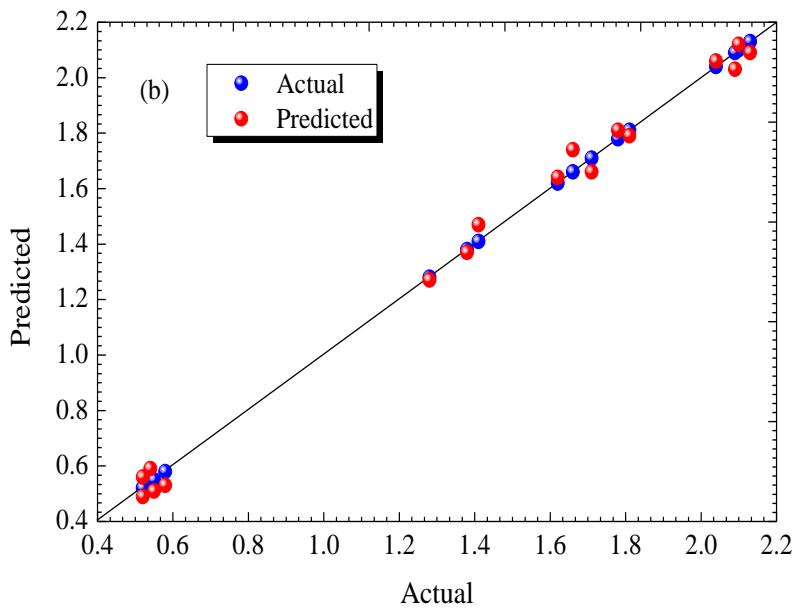
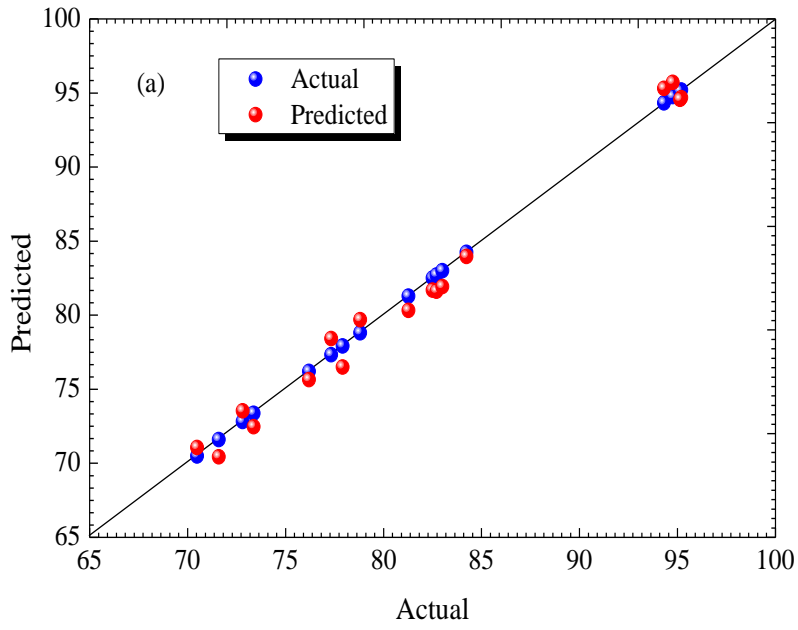


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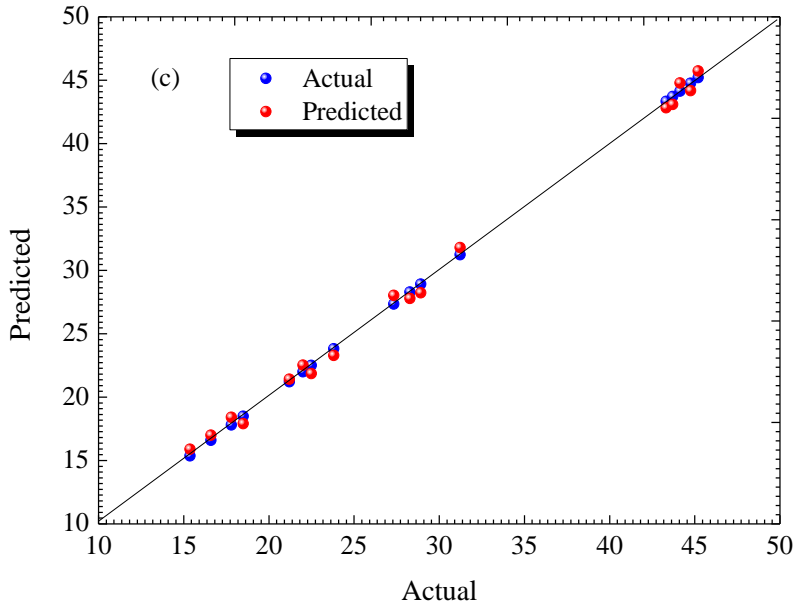


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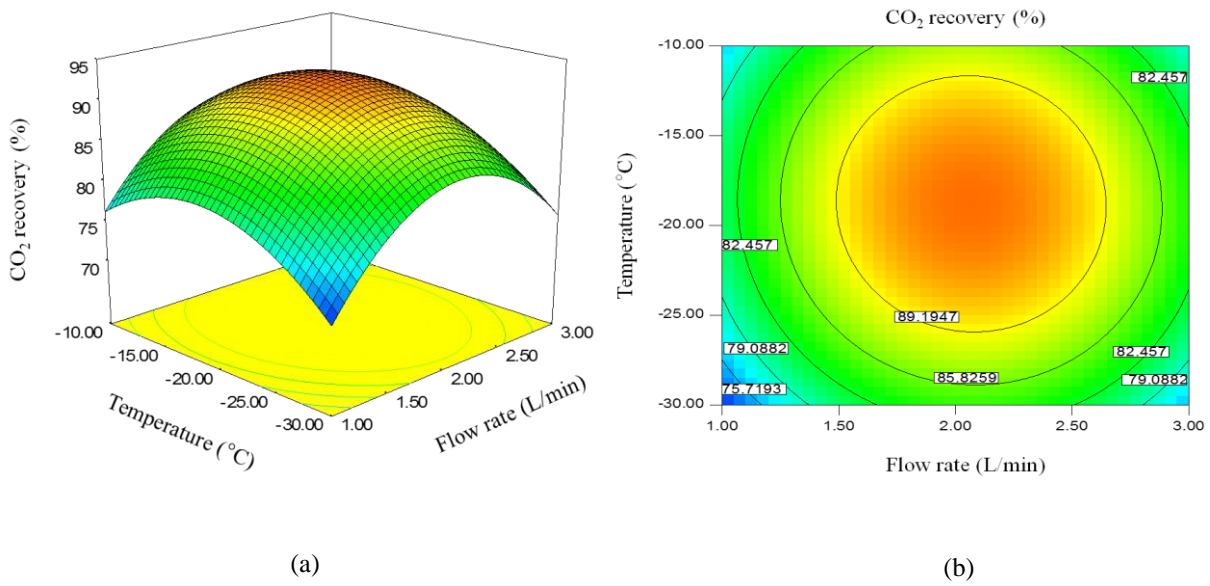
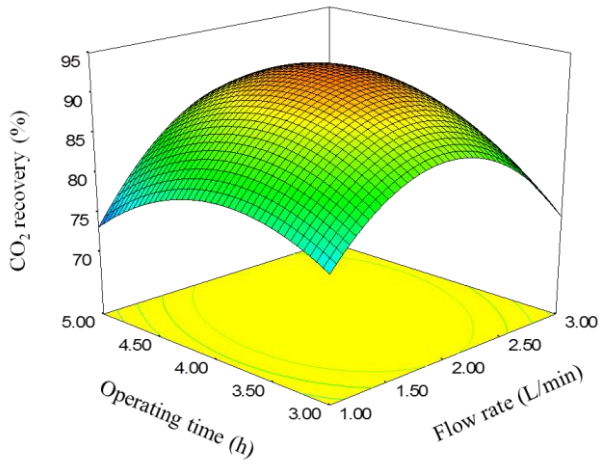
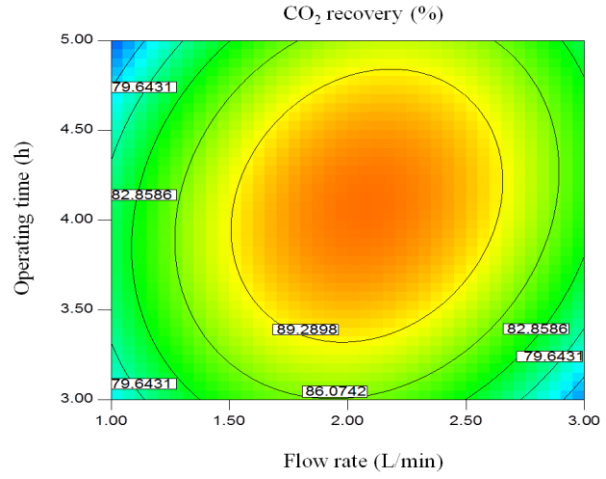


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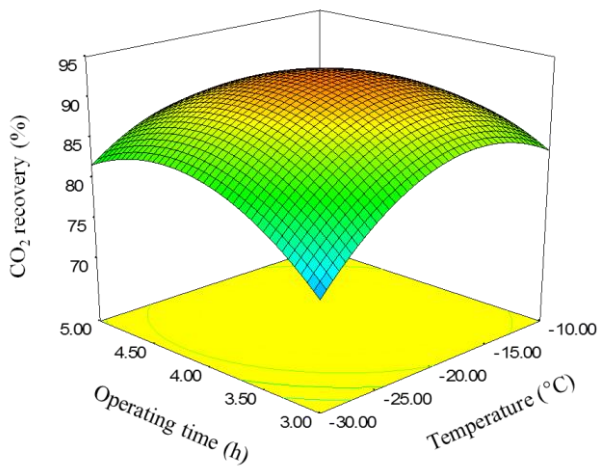


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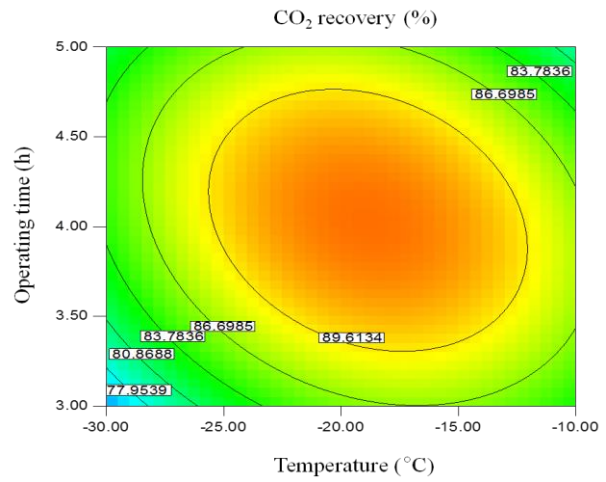


(b)

Fig. 5 Interaction effect of flow rate ( $X_1$ ) and operating time ( $X_3$ ) on  $\text{CO}_2$  recovery (%). (a) 3D surface plot; (b) contour plot



(a)



(b)

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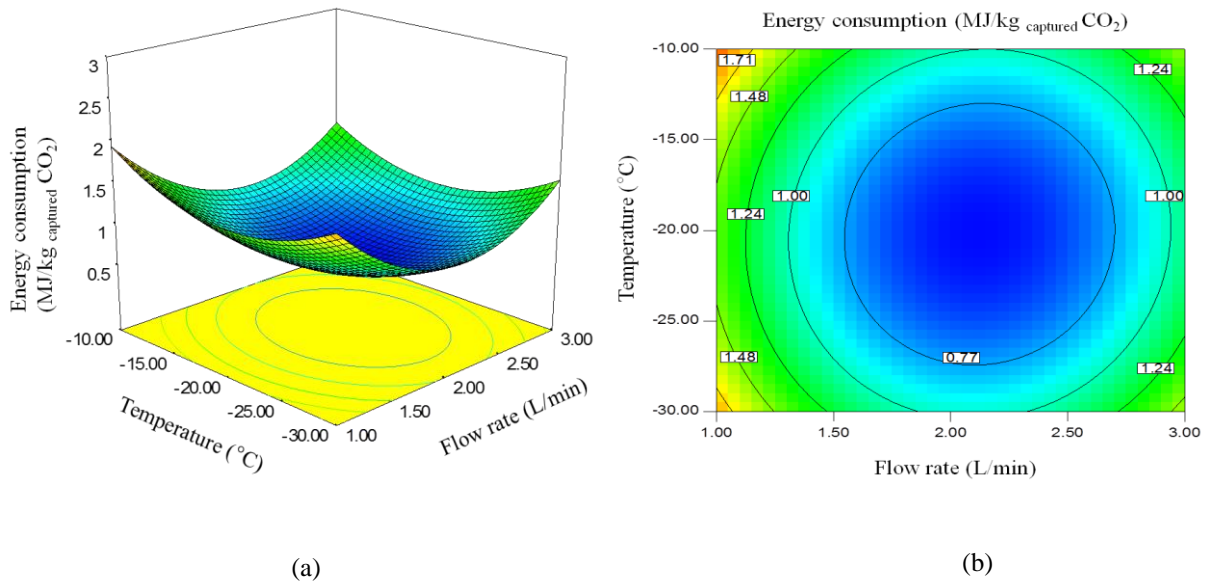


Fig. 7 Interaction effect of flow rate ( $X_1$ ) and temperature of FPSC-1 ( $X_2$ ) on energy consumption ( $\text{MJ/kg}_{\text{captured CO}_2}$ ).

(a) 3D surface plot; (b) contour plot

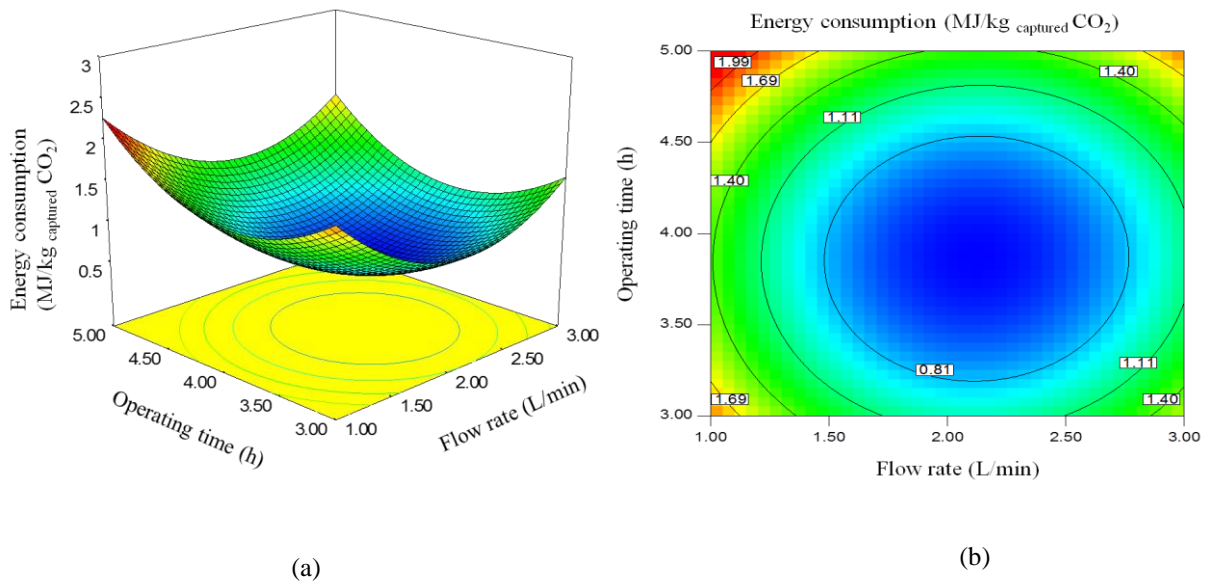
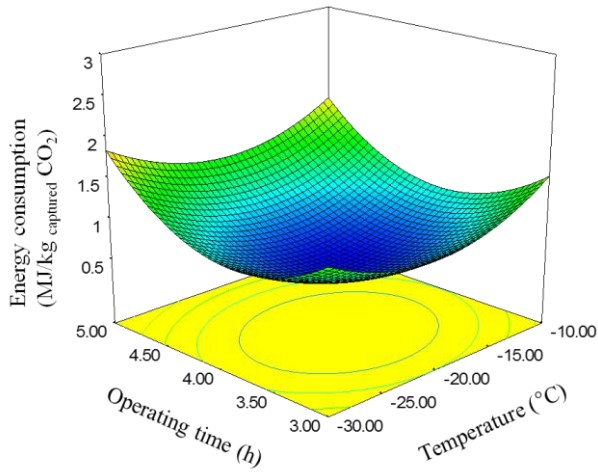
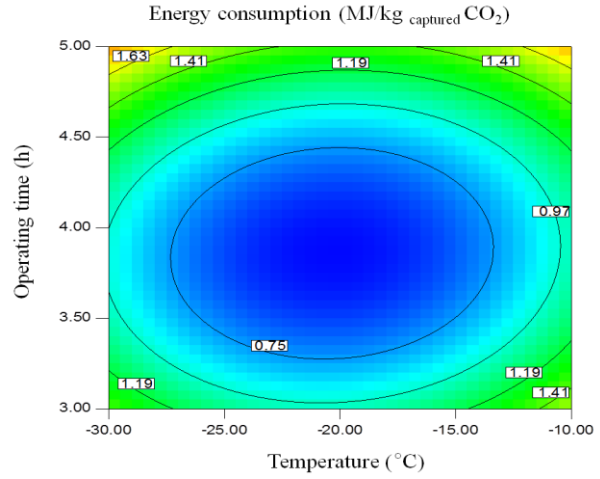


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(a) 3D surface plot; (b) contour plot

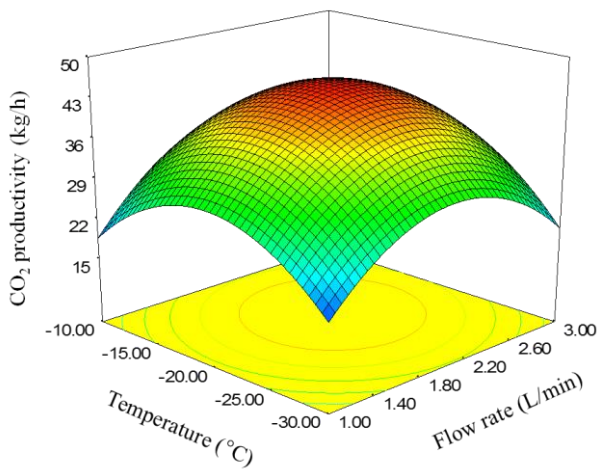


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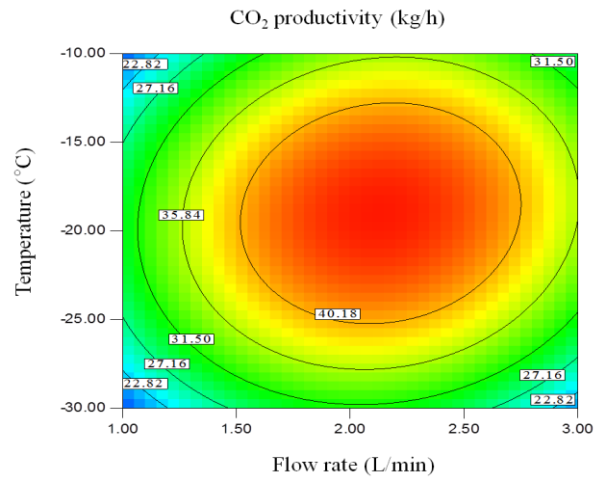


(b)

Fig. 9 Interaction effect of temperature of FPSC-1 ( $X_2$ ) and operating time ( $X_3$ ) on energy consumption (MJ/kg captured CO<sub>2</sub>). (a) 3D surface plot; (b) contour plot



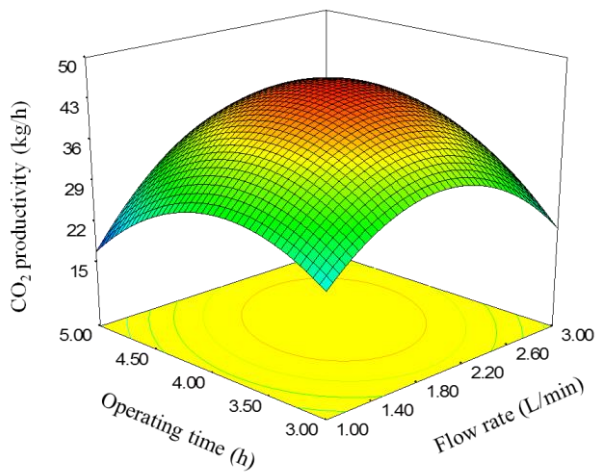
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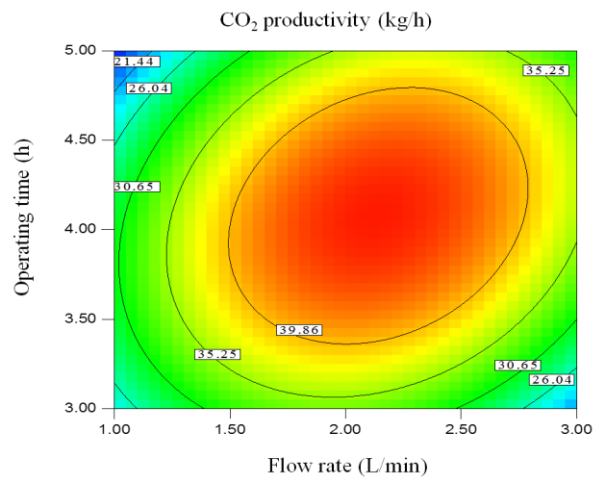
(b)

Fig. 10 Interaction effect of flow rate ( $X_1$ ) and temperature of FPSC-1 ( $X_2$ ) on CO<sub>2</sub> productivity (kg/h). (a) 3D surface plot; (b) contour plot



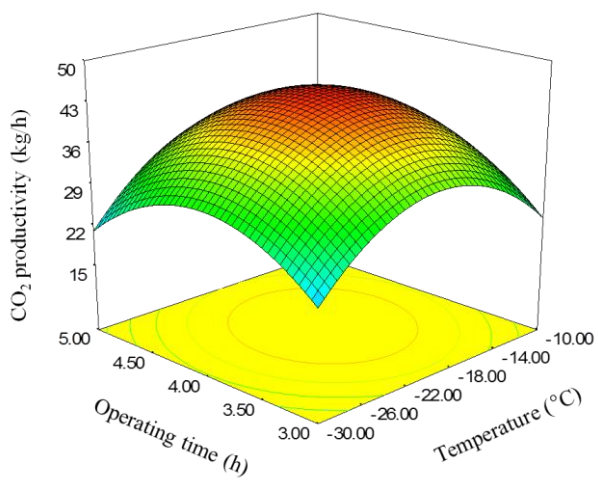


(a)

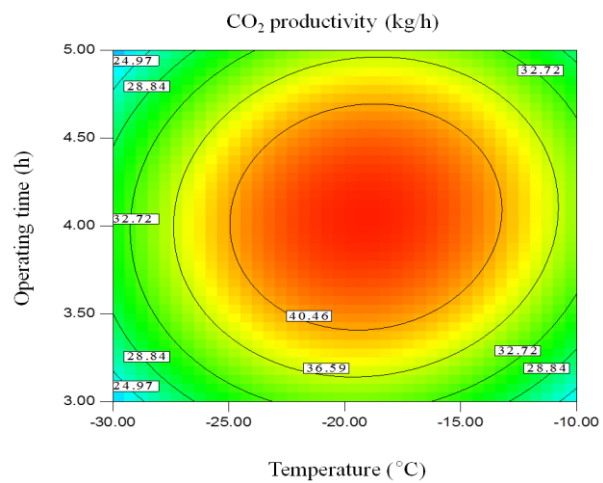


(b)

Fig. 11 Interaction effect of flow rate ( $X_1$ ) and operating time ( $X_3$ ) on  $\text{CO}_2$  productivity (kg/h). (a) 3D surface plot; (b) contour plot



(a)



(b)

Fig. 12 Interaction effect of temperature of FPSC-1 ( $X_2$ ) and operating time ( $X_3$ ) on  $\text{CO}_2$  productivity (kg/h). (a) 3D surface plot; (b) contour plot

## Table captions

**Table 1** Independent variables and their levels.

**Table 2** Experimental matrix for CCD during the capture process.

**Table 3** ANOVA analysis for CO<sub>2</sub> recovery ( $\eta$ ) model.

**Table 4** ANOVA analysis for energy consumption (EC) model.

**Table 5** ANOVA analysis for CO<sub>2</sub> productivity ( $\varphi$ ) model.

**Table 6** Results of validation test.

**Table 1**

Independent variables and their levels.

Factors	Tag	Symbol	Units	level		
				-1	0	1
Flow rate of flue gas	v	X <sub>1</sub>	L/min	1	2	3
Temperature of FPSC-1	T	X <sub>2</sub>	°C	-30	-20	-10
Idle operating time	t	X <sub>3</sub>	h	3	4	5

**Table 2**

Experimental matrix for CCD during the capture process.

Run no.	Experimental information						Results					
	X <sub>1</sub> (v-L/min)		X <sub>2</sub> (T-°C)		X <sub>3</sub> (t-h)		Y <sub>1</sub> (η-%)		Y <sub>2</sub> (EC-MJ/kg <sub>captured</sub> CO <sub>2</sub> )		Y <sub>3</sub> (φ-kg/ h)	
	Coded	Actual	Coded	Actual	Coded	Actual	Experimental	Predicted	Experimental	Predicted	Experimental	Predicted
1	0	2	-1	-30	-1	3	70.49	71.06	1.38	1.37	18.49	17.89
2	1	3	0	-20	-1	3	76.20	75.64	1.71	1.66	21.20	21.40
3	-1	1	0	-20	-1	3	81.28	80.32	1.66	1.74	28.29	27.79
4	-1	1	0	-20	1	5	71.60	70.44	2.13	2.09	16.60	16.99
5	0	2	0	-20	0	4	94.32	95.31	0.54	0.56	43.33	42.83
6	0	2	0	-20	0	4	95.20	94.67	0.52	0.53	45.20	45.73
7	0	2	0	-20	0	4	94.76	95.71	0.55	0.51	44.76	44.18
8	0	2	1	-10	-1	3	82.50	81.68	1.62	1.64	22.50	21.86
9	1	3	1	-10	0	4	77.92	76.5	1.28	1.27	28.92	28.22
10	0	2	0	-20	0	4	82.70	81.61	0.58	0.53	43.70	43.09
11	-1	1	-1	-30	0	4	72.81	73.53	2.04	2.06	17.81	18.41
12	1	3	0	-20	1	5	77.33	78.42	2.10	2.12	27.33	28.02
13	1	3	-1	-30	0	4	78.81	79.69	1.41	1.47	23.81	23.30
14	-1	1	1	-10	0	4	73.37	72.47	2.09	2.03	15.37	15.89
15	0	2	1	-10	1	5	84.23	83.96	1.81	1.79	31.23	31.80
16	0	2	-1	-30	1	5	83.00	81.92	1.78	1.81	22.00	22.52
17	0	2	0	-20	0	4	95.13	94.57	0.52	0.53	44.13	44.77

**Table 3**ANOVA analysis for CO<sub>2</sub> recovery ( $\eta$ ) model.

Factors	Sum of squares	df	Mean of squares	F-value	P-value	Significance
Model	61.81	9	6.87	44.65	<0.0001	significant
X <sub>1</sub>	49.01	1	49.01	318.63	<0.0001	
X <sub>2</sub>	1.20	1	1.20	7.78	0.0270	
X <sub>3</sub>	2.74	1	2.74	17.82	0.0039	
X <sub>1</sub> <sup>2</sup>	2.43	1	2.43	15.80	0.0054	
X <sub>2</sub> <sup>2</sup>	0.89	1	0.89	5.79	0.0471	
X <sub>3</sub> <sup>2</sup>	0.035	1	0.035	0.22	0.6500	
X <sub>1</sub> X <sub>2</sub>	0.52	1	0.52	3.40	0.1079	
X <sub>1</sub> X <sub>3</sub>	4.37	1	4.37	28.43	0.0011	
X <sub>2</sub> X <sub>3</sub>	0.38	1	0.38	2.45	0.1615	
Residual	1.08	7	0.15			
Lack of Fit	0.85	3	0.28	4.97	0.0777	not significant
Pure Error	0.23	4	0.057			
Cor Total	62.89	16				
Standard deviation = 0.39				R <sup>2</sup> = 0.9829		
Mean = 80.12				Adj R <sup>2</sup> = 0.9609		
Coefficient of variation = 0.49				Pred R <sup>2</sup> = 0.7784		
Predicted residual error of sum of squares = 13.94				Adeq Precision = 23.409		

**Table 4**

ANOVA analysis for energy consumption (EC) model.

Factors	Sum of squares	df	Mean of squares	F-value	P-value	Significance
Model	3.82	9	0.42	53.64	<0.001	significant
X <sub>1</sub>	3.02	1	3.02	381.54	<0.0001	
X <sub>2</sub>	8.210E-003	1	8.210E-003	1.04	0.3423	
X <sub>3</sub>	0.18	1	0.18	22.92	0.0020	
X <sub>1</sub> <sup>2</sup>	0.59	1	0.59	74.76	0.6933	
X <sub>2</sub> <sup>2</sup>	0.020	1	0.020	2.53	0.6415	
X <sub>3</sub> <sup>2</sup>	1.946E-005	1	1.946E-005	2.460E-003	0.4558	
X <sub>1</sub> X <sub>2</sub>	1.337E-003	1	1.337E-003	0.17	<0.0001	
X <sub>1</sub> X <sub>3</sub>	1.872E-003	1	1.872E-003	0.24	0.1559	
X <sub>2</sub> X <sub>3</sub>	4.930E-003	1	4.930E-003	0.62	0.9618	
Residual	0.055	7	7.912E-003			
Lack of Fit	9.864E-003	3	3.288E-003	0.29	0.8322	not significant
Pure Error	0.046	4	0.011			
Cor Total	5.86	19				
Standard deviation = 0.089				R <sup>2</sup> = 0.9857		
Mean = 1.07				Adj R <sup>2</sup> = 0.9673		
Coefficient of variation = 8.31				Pred R <sup>2</sup> = 0.9409		
Predicted residual error of sum of squares = 0.23				Adeq Precision = 22.423		

**Table 5**ANOVA analysis for CO<sub>2</sub> productivity ( $\phi$ ) model.

Factors	Sum of squares	df	Mean of squares	F-value	P-value	Significance
Model	36.34	9	6.04	16.33	<0.001	significant
X <sub>1</sub>	7.27	1	7.27	21.39	<0.0001	
X <sub>2</sub>	31.63	1	31.63	237.53	0.1556	
X <sub>3</sub>	5.58	1	5.58	0.45	0.5252	
X <sub>1</sub> <sup>2</sup>	5.45	1	5.45	24.39	0.0003	
X <sub>2</sub> <sup>2</sup>	5.28	1	5.28	24.85	0.0003	
X <sub>3</sub> <sup>2</sup>	3.71	1	3.71	9.75	0.0010	
X <sub>1</sub> X <sub>2</sub>	1.427	1	1.427	3.14	0.3207	
X <sub>1</sub> X <sub>3</sub>	7.937	1	7.937	29.35	0.0398	
X <sub>2</sub> X <sub>3</sub>	6.83	1	6.83	0.55	0.4836	
Residual	1.87	7	0.13			
Lack of Fit	0.61	3	0.18	3.85	0.0513	not significant
Pure Error	0.32	4	0.058			
Cor Total	29.24	16				
Standard deviation = 0.35				R <sup>2</sup> = 0.9545		
Mean = 29.10				Adj R <sup>2</sup> = 0.8961		
Coefficient of variation = 12.15				Pred R <sup>2</sup> = 0.7902		
Predicted residual error of sum of squares = 36.55				Adeq Precision = 20.104		

**Table 6**

Results of validation test.

Run no.	X <sub>1</sub> (v-L/min)	X <sub>2</sub> (T-°C)	X <sub>3</sub> (t-h)		Y <sub>1</sub> (η-%)	Y <sub>2</sub> (EC-MJ/kg <sub>captured</sub> CO <sub>2</sub> )	Y <sub>3</sub> (φ-kg/h)
1	1.5	-15	3.5	Experimental	81.14	1.23	21.83
				Predicted	80.34	1.21	21.45
				Error (%)	0.99	1.62	1.74
2	2.0	-20	4.0	Experimental	95.45	0.56	44.17
				Predicted	94.81	0.52	43.22
				Error (%)	1.33	3.92	2.15
3	2.5	-25	4.5	Experimental	78.09	1.48	33.94
				Predicted	78.83	1.51	33.51
				Error (%)	0.95	2.03	1.27