1	Effect of shear flow on the hydrodynamic drag force of a spherical
2	particle near wall evaluated using optical tweezers and microfluidics
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#### 19 Abstract

The hydrodynamic drag force on a spherical particle in shear flow near-wall is 20 21 investigated using optical tweezers and microfluidics. Simple shear flow is applied using a microfluidic channel at different volumetric flow rates. The hydrodynamic drag 22 force exerted on the particle is detected from the displacement of the trapped particle. 23 The effect of the wall is obtained from the force balance of the trapping and 24 hydrodynamic drag force employing the exact solution of the theoretical model using 25 the lubrication theory for a sphere near the wall. Here, we report the experimentally 26 27 obtained hydrodynamic drag force coefficient under the influence of shear flow. The drag correction factor increases with decreasing distance from the wall due to the effect 28 29 of the wall surface. We found that the calculated hydrodynamic drag force coefficient 30 is in quantitative comparison with the theoretical prediction for a shear flow past a sphere near-wall. This study provides a straightforward investigation of the effect of the 31 shear flow on the hydrodynamic drag force coefficient on a particle near the wall. 32 33 Furthermore, these pieces of information can be used in various applications, particularly in optimizing microfluidic designs for mixing and separations of particles 34 or exploit the formation of the concentration gradient of particles perpendicular to flow 35 36 directions caused by the non-linear hydrodynamic interactions

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Keyword: Hydrodynamic shear flow/Optical tweezers/microfluidics/drag force

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## 40 **1. Introduction**

Flow of colloidal suspension is a fundamentally and practically essential 41 42 process that is present in a plethora of applications. The flow of colloidal particles in rivers,<sup>1,2</sup> transport of particles in filtration,<sup>3</sup> microfluidic flows in biomedical 43 devices,<sup>4,5</sup> and the flow of blood cells in the body, to name a few, are the 44 important natural and technological settings of the flow of colloids. Common to 45 all these applications, adhesion or deposition<sup>6-10</sup> of colloidal particles onto the 46 surface in contact with the flow is one of the most critical issues-for example, 47 adhesion of cells to tissues in bloodstreams is essential to many 48 pathophysiological processes,<sup>11,12</sup> or the "tubular pinch" effect of diffusing 49 spherical particle in Poiseuille flow along a narrow gap annulus is vital in 50 membrane technology.<sup>13,14</sup> Even though the kinetics of deposition of colloidal 51 particles is influenced by several factors like flow conditions, surface chemistry, 52 and particle volume fraction, it is widely recognized that particle-wall 53 54 interactions generally define the system. Hence, understanding the effect of hydrodynamic forces on a particle in the vicinity of a wall translating by a shear 55 flow is of fundamental importance. 56

Accordingly, several numerical models based on fluid dynamics have been proposed to estimate the forces and torques on particles translating near-wall. Dean and O'Neill<sup>15</sup> addressed the issue of the flow around a rotating sphere nearwall using bipolar coordinates. Moreover, O'Neill<sup>16</sup> explicitly determined the

hydrodynamic forces about the fixed sphere in contact with fluid motion using 61 the exact solution of the linearized Stokes flow equations. These issues are 62 revisited by Goldman et al.,17,18 incorporating the lubrication theory for an 63 64 extended analysis in the case wherein the gap between the particle and wall is smaller than the particle radius in a quiescent and Couette flow. O'Neill and 65 Stewartson<sup>19,20</sup> further reconsidered the lubrication problem by matched 66 asymptotic expansions and Perkins and Jones<sup>21</sup> using the terms of the Green 67 function for bounded fluid. Additionally, Magnaudet et al.<sup>22</sup> evaluated the slip 68 and shear effects in wall-bounded flows to infer the hydrodynamic drag on a 69 fixed particle. Later, Chaoui and Feuiollebois<sup>23</sup> re-examined the creeping flow 70 71 around a sphere using bipolar coordinates with improved numerical calculations, wherein the earlier results were recovered with a precision order of  $10^{-11}$ . 72 On one side, a large number of numerical simulations have been performed 73

to verify the hydrodynamic drag force on the flowing particle near-wall.<sup>24-30</sup> Zeng 74 et al.<sup>25</sup> performed direct numerical investigations on the hydrodynamic drag 75 76 force on a particle in a linear shear flow with finite slip at near wall. Recently, Ekanayake *et al.*<sup>30</sup> numerically evaluated the drag and lift forces on a particle 77 near the wall in shear flow with an improved wall-shear-based drag correlation. 78 79 The method accurately captures the drag force and follows the results of the theoretical models. However, experimental studies evaluating the hydrodynamic 80 drag force on the particle under the influence of shear flow are relatively scarce 81 82 due to the difficulty in determining the actual forces in situ. Although some methods are utilized to understand the hydrodynamic drag force of the particle near-wall, such as studies on hydrodynamic interactions in quiescent or uniform flow conditions,<sup>31-33, 49-50</sup> to the best of our knowledge, no detailed investigations are performed to evaluate the hydrodynamic drag force exerted on a particle near a wall under shear flow. Hence the motivation of this study is a direct evaluation of the hydrodynamic force on a particle near a wall under shear flow, which is very important in a plethora of applications.

90 Optical tweezers have become an essential tool in various fields due to their versatile applications.<sup>34-40</sup> With the advent of optical trapping and microfluidics, 91 92 diverse applications ranging from chemistry and physics to medicine and biology have become available. One of the most emerging lab-on-a-chip device 93 applications controls and manipulates colloidal particles under varying flow 94 fields and solution environments. From this standpoint, a micrometer-sized 95 particle can be trapped, and the displacement of the particle in the trap is a 96 measure of the force exerted on it. Hence the corresponding forces felt by the 97 98 trapped particle can be exploited to characterize the surrounding environment.<sup>39,40</sup> Additionally, with the increasing applications of microfluidic 99 100 devices, it is important to evaluate the effect of these hydrodynamic interactions 101 between particle and wall. Precise control over particle position in microfluidic devices allows for more efficient and high throughput operations on flowing 102 objects in microchannels. Therefore, with optical trapping and precise control of 103 104 fluid flow using microfluidics, it is ideally possible to directly evaluate the

105 hydrodynamic drag force exerted on a particle under shear flow at any location106 within the channel.

107 Therefore, this study aimed to investigate the hydrodynamic drag force on a 108 colloidal particle within the vicinity of a near-wall under the influence of shear 109 flow using optical tweezers and microfluidics. Even though this is a classical problem, to the best of our knowledge, a quantitative comparison between 110 theoretical and experimental measurements has not been reported to date. A 111 112 significant advantage of this study stems from its ability to directly measure the total force acting on the particle under shear flow. Furthermore, back-focal-plane 113 114 detection in optical tweezers provides high temporal and spatial resolution for the force measurements. Hence, we report on the experimentally obtained drag 115 correction factor and compare it with the theoretical model and numerical 116 calculations on the particle-wall interaction under shear flow to further extend 117 the previous measurements of the translational drag coefficient. 118

119 2. Materials and Methods

## 120 2.1 Experimental setup

121 An optical trapping kit (OTKB/M, Thorlabs) equipped with a single laser 122 (wavelength  $\lambda$ =976 nm) was used in the experiments. The laser light was 123 delivered to the system via an SM980-5.8-125 single-mode fiber (Thorlabs). A 124 100× oil immersion objective with a high numerical aperture (NA 1.25, WD 0.23 125 mm, Nikon) was used to tightly focus the laser beam. An air condenser (10×, NA

0.25, WD 7 mm, Nikon) was used to collect the laser light passing through the 126 sample and further reflected into the quadrant position detector (QPD) using a 127 dichroic mirror for the back-focal plane detection (OTKBFM, Thorlabs). The 128 129 QPD was connected to the force measurement module (OTKBFM-CAL, 130 Thorlabs) for data acquisition. The time-series signals of the x- and y-131 displacement of the particle in the trap were recorded at a rate of 10 kHz for at least ten seconds using the software package for the OTKBFM-CAL Force 132 Acquisition Module (OTKB-Cal, Thorlabs). A piezo-controlled 3-dimension 133 translational stage (NanoMax 300, Thorlabs) was used to position the micro-134 135 channel with respect to the optical trap using the Thorlabs APT software package. The particle position was controlled using the piezo-controlled 3-dimensional 136 stage in stepwise increments with a readout resolution of  $0.32\pm0.02 \ \mu m/V$ . The 137 axial distance of the particle relative to the surface of the channel bottom, h/a, 138 where h is the axial distance of the centroid of the particle from the bottom 139 surface of the inner wall, and *a* is the particle radius, was calibrated manually by 140 141 moving the stage. In this way, the axial distance h was controlled while the position of the particle was kept constant using the optical trap. 142

The voltage fluctuation of the trapped particle was converted into particle displacement using the QPD calibration factor. The voltage-to-position response function of the QPD was calibrated by displacing the particle at the equilibrium position in the trap using fluid flow. The constant particle displacement was measured by image processing, and the QPD signal was recorded. The slope of particle displacement (µm) as a function of QPD response (V) was determined to
be 0.48 V/µm (see Supplementary Fig. 1).

The probe particles used in the experiments were non-functionalized silica 150 151 (diameter 2a=2.5 µm, Nippon Shokubai, Japan) suspended in pure water. Silica particles are negatively charged in aquatic solution <sup>41,42,</sup> and the double layer 152 repulsion reduces the adhesion of silica to a glass surface. Particle suspensions 153 were prepared by diluting the stock suspension of 0.1 wt.% to 0.0001 wt.%. 154 155 Deionized water (Elix Advantage 5, Millipore, Tokyo, Japan) was used to prepare all particle suspensions. All the experiments were conducted at room 156 157 temperature (20 °C).

## 158 **2.2 Flow measurements**

Flow experiments were performed using a parallel flow chamber (sticky-Slide I, 159 0.1 Luer, Part of: 81128, Ibidi, Germany) with channel dimensions (length L=48000 160  $\mu$ m, width W=5000  $\mu$ m, and total height H=150  $\mu$ m, including the double sticking tape). 161 162 The glass coverslip (#1, 0.12~0.17 mm, NEO, Matsunami, Japan) was attached to the flow cells by double sticking tape (thickness  $=50 \mu m$ ). Coverslips were used as 163 provided by the manufacturer without any surface modifications. The flow chamber 164 and coverslip were tightly compressed and incubated at 40 °C overnight to obtain strong 165 adhesion, following the recommendation of the manufacturer (Ibidi, Germany). The 166 microchannel was connected to a 1 mL plastic syringe (inner diameter =4.65 mm, Soft-167 168 ject, Henke-Sass Wolf, Germany) using silicone tubing. Particle suspensions were injected into the flow cell using a high-precision syringe pump (Nexus 3000, Chemyx Inc., USA). A random particle was trapped at a fixed laser current of 250 mA. After that, pure water was flowed into the channel to flush the remaining particles to remove interparticle interaction during the measurements. The flow measurements were performed at different volumetric flow rates at 500  $\mu$ L/hr, 1000  $\mu$ L/hr, and 1500  $\mu$ L/hr, respectively. The displacement of the particle at various *h/a* with respect to the equilibrium position was recorded for at least ten seconds.

176 **2.3 Stiffness calibration** 

The position at which the particle touched the surface (h/a = 1) was determined from the drastic change in the QPD signal.<sup>30,41</sup>After determining the reference position, h/a= 1, the particle position was then changed to  $h/a \sim 10$  by moving the stage. After that, the particle fluctuation was recorded for at least ten seconds, and the corresponding trap stiffness was determined using the power spectral density (PSD) roll-off method.

The optical trap stiffness was calibrated at various h/a, as shown in Fig. 1. The laser current was set to 250 mA for the trapping. The measured trap stiffness k showed a weakening with increasing h/a. This dependence is expected for an oil-immersion objective due to spherical aberrations caused by the refractive index mismatch, which agrees with the previous results.<sup>37,43</sup> Therefore, the fitting line in Fig. 1 is used to obtain the actual trap stiffness k(h/a).

188 **3. Results and Discussion** 

#### 189 **3.1 Hydrodynamic effect near-wall at the quiescent condition**

The hydrodynamic effect on the particle near the wall is investigated by following the change in the thermal fluctuation of the trapped particle as it is brought closer to the wall. Fig. 2a presents the representative 1-dimensional timeseries fluctuation of an optically trapped particle without shear flow at representative h/a. An apparent attenuation in the thermal fluctuation could be observed as the particle approaches the wall due to the hydrodynamic effect.

The thermal fluctuation of the trapped particle is analyzed to quantify the change in the hydrodynamic drag using the normalized position autocorrelation function (NPAF),<sup>40, 51</sup> described as follows:

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$$A(\tau) = \frac{\langle \vec{x}(t)\vec{x}(t+\tau)\rangle_t}{\langle x^2 \rangle_t}$$
(1)

where  $\vec{x}(t)$  is the particle position at time *t*,  $\tau$  is the time interval (lag-time) a nd  $\langle x^2 \rangle$ is the time-independent variance of the particle. Based on the assumption of a sufficiently long measurement (i.e., larger than  $1/\lambda$ ) of the thermal fluctuation of the particle in a Newtonian fluid with constant viscosity, the NPAF can be approximated by a single exponential decay<sup>40, 51</sup>

205  $A(\tau) = e^{-\lambda \tau}$ (2)

where  $\lambda = k/\gamma$ , *k* is the trap stiffness, and  $\gamma = 6\pi a\eta$  is the drag coefficient where  $\eta$  is the fluid viscosity. Correspondingly, the characteristic  $\lambda$  is directly related to the relaxation rate of the system, also known as the *corner frequency*, when the particle fluctuation is analyzed in terms of the power spectral density.<sup>40,44,46</sup>

210 The normalized  $\gamma/\gamma_0$  as a function of h/a is shown in Fig. 2b. The measured drag 211 coefficient  $\gamma$  is normalized using the value of  $\gamma$  farthest from the surface,  $\gamma_0$ . The data

for  $\lambda_0/\lambda$  is also shown for reference. Since the k varies with h/a, the corresponding  $\gamma$  is 212 calculated using the actual k(h/a) obtained in the calibration. As Fig. 2b shows, the  $\gamma/\gamma_0$ 213 increases dramatically at a distance close to the wall because of the hydrodynamic 214 215 friction, and at the same time, it is constant at approximately large distances. The 216 normalized  $\gamma/\gamma_0$  is compared with the theory for a particle moving parallel to the wall by Goldman et al.<sup>17</sup> and Faxen's correction for lateral directions. The results are in good 217 agreement and consistent with the expectations from the hydrodynamic theory. Hence, 218 219 the attenuation of the thermal fluctuation of the trapped particle as it is brought closer to the wall is due to the hydrodynamic interactions between the particle and the surface. 220 Accordingly, these results were consistent with the previous studies, 35,47,49-50 221 confirming the applicability of our methodology to investigate the hydrodynamic 222 interactions between the particle within the vicinity of a wall in a more complicated 223 scenario such as under shear flow conditions. 224

#### 225

## 3.2 Hydrodynamic effect near-wall by shear flow

The more complex hydrodynamic interactions between a spherical particle and 226 227 wall under simple shear flow are evaluated. Simple shear flow is obtained from the gradient of the fluid velocity close to the wall. Considering no-slip boundary conditions, 228 the fluid velocity profile in a rectangular channel is accessible as an analytical solution 229 of the Navier-Stokes equation for a pressure-driven flow. The fluid velocity in the x-230 direction can be expressed as follows:<sup>48, 52</sup> 231

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$$U_{x}(y,h) = \frac{4H^{2}\Delta p}{\pi^{3}\eta L} \sum_{n,odd}^{\infty} \frac{1}{n^{3}} \left[ 1 - \frac{\cosh\left(n\pi\frac{y}{H}\right)}{\cosh\left(n\pi\frac{W}{2H}\right)} \right] \sin\left(n\pi\frac{yh}{H}\right)$$
(3)

with *H*, *W*, *L*, and  $\eta$  as previously defined, and  $\Delta p/L$  is the pressure gradient. The pressure gradient  $\Delta p/L$  is further defined in terms of the volumetric flow rate, *Q*, given as follows: <sup>48, 52</sup>

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$$Q = \frac{H^3 W \Delta p}{12 \eta L} \left[ 1 - \sum_{n,odd}^{\infty} \frac{1}{n^5} \frac{192}{\pi^5} \frac{H}{W} \tanh(n\pi \frac{W}{2H}) \right].$$
(4)

237 The theoretical fluid velocity within the channel is plotted in Fig. 3. At distances close 238 to the channel walls, the fluid velocity increases almost linearly; thereby, it is defined as a simple shear flow. The calculated fluid velocity at different h/a was also confirmed 239 experimentally at a volumetric flow rate of 500 µL/hr by tracking the flowing particle 240 after being released in the trap, as shown in the open symbol in Fig. 3. A good agreement 241 242 between the predicted and experimentally obtained local fluid velocity is shown. 243 Additionally, due to the relatively thick microfluidic channel in comparison to the working distance of the optical trap, the measurement was performed at the limited 244 axial range of h/a. Nevertheless, the fluid velocity linearly increases within the 245 observation range of h/a; therefore, this method can be utilized to apply a simple shear 246 flow on a particle within the vicinity of the wall. 247

Fig. 4 presents the displacement of the trapped particle,  $\Delta x$ , from its equilibrium position at various h/a for different volumetric flow rates Q. In general, when an external force is applied to the trapped particle, the particle displacement from the equilibrium point is proportional to the applied force. Also, it has been shown that the optical trap can be treated approximately as a linear spring where the force  $F_{\text{trap}}$ , is given by  $F_{\text{trap}} = k\Delta x$  where k is previously defined.

In Fig. 4, the  $\Delta x$  increased with increasing h/a for all volumetric flow rates. This

increase corresponds to the applied hydrodynamic force by shear flow. Based on the linearity of Stokes' equation, the external force acting on the trapped particle is due to the hydrodynamic resistance force (drag force). For an unbounded fluid flow or at large h/a, the drag force is only a function of fluid velocity. However, for a particle translating near the wall, the drag force exerted on the particle is affected by the presence of the boundary surfaces, which nonlinearly modifies the hydrodynamic drag exerted by the fluid.

262 The analytical solution of the hydrodynamic drag force exerted on a particle near 263 wall in a simple shear flow is expressed as follows:<sup>18</sup>

$$F_d = 6\pi a\eta U_x f^*(h/a) \tag{5}$$

where *a* and  $\eta$  is previously defined,  $U_x$  is the fluid velocity and  $f^*(h/a)$  is the correction factor of the drag friction coefficient, which is a function of *h/a*. The  $f^*(h/a)$  in Eq. 5 is considered highest when the particle is in contact with the wall and decays rapidly to the unbounded value as the *h/a* increases.<sup>18</sup> Using the exact value of  $U_x$  at *h/a*, the  $f^*(h/a)$  was calculated by taking the force balance between the trapping force and the effective drag force wherein the total force acting on the particle is given as follows:

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$$k\Delta x = 6\pi a\eta U_x f^*(h/a). \tag{6}$$

Fig. 5 presents the  $f^*$  as a function of h/a between the particle and the wall. Also plotted for comparison in the Figure as a solid line is the exact solution of the theoretical model obtained by Goldman *et al.*<sup>18</sup> From Fig. 5, it is clear that the drag force increases with decreasing h/a between the particle and the wall. When the particle is near the vicinity of the wall, the space between the particle and the surrounding fluid is reduced; hence, increasing the corresponding drag force in a direction parallel to the wall. Therefore, the total drag force experienced by the particle is observed to be increased by the presence of the wall, while beyond that, the particle experiences an unbounded fluid flow. The experimentally calculated  $f^*$  over the range of h/a is in good agreement with the theoretical values of Goldman *et al.*,<sup>18</sup> verifying the applicability of the theoretical model.

Furthermore, the hydrodynamic drag force coefficient did not show any 284 dependency for the studied volumetric flow rates. Based on the force balance equation 285 286 between the trapping and drag force in Eq. 6, the force felt by the particle is only a function of the fluid velocity at a certain distance, which is proportional to the 287 displacement in the optical trapping force. Thereby, the volumetric flow rate increase 288 would only mean an increase in the applied drag force at laminar flow conditions and, 289 consequently, the displacement  $\Delta x$  as shown in Fig. 3. This reflects the independence 290 of the calculated hydrodynamic drag coefficient on the volumetric flow rate, at least at 291 292 laminar flow conditions. Moreover, this independence also corroborates the assumption of the theoretical model wherein no-slippage, Newtonian fluid flow, and low Reynold's 293 number conditions are satisfied. However, for conditions where slippage, non-294 Newtonian fluid, or large Reynold's number cases is considered, this independence 295 might be affected.<sup>53</sup> 296

Although it is well known that shear flow on a particle near the wall can also induce torque and lift force,<sup>18, 25,30</sup> such effects are not considered in the present analysis. For

a spherical particle made of homogenous, transparent isotropic material, optical torque 299 300 is not considered; thus, the particle rotation and torque are not considered in the present analysis. Meanwhile, the lift force is assumed nullified by the optical trapping force, 301 302 limiting the axial displacement of the particle while under the influence of shear flow. 303 Thus, here we only ponder that the drag force influences the particle dynamics predominantly. Moreover, Goldman et al.<sup>18</sup> independently considered the effect of 304 hydrodynamic drag and toque. Therefore, Eq. 5 is derived solely for the hydrodynamic 305 drag force exerted by the fluid on the particle. However, the effect of torque and lift 306 force is similarly important and should be considered in future research, especially in 307 non-steady flow conditions and in non-Newtonian fluids.<sup>53</sup> 308

It is also important to compare our results with the earlier study of Eom et al.<sup>45</sup> 309 using optical tweezers to probe the fluid flow. However, the experimental range of their 310 study was a lot larger than the present study; thus, the effect of near-wall was not 311 accounted for in their analysis. Nevertheless, the technique using optical tweezers and 312 microfluidics provides the direct force measurement on the hydrodynamic drag force 313 314 exerted on a particle near-wall under shear flow. From an applied perspective, understanding the hydrodynamic interactions in different flow conditions is very 315 important, particularly in microfluidics applications, where colloids are used for various 316 applications. Since most studies focus on the hydrodynamic interactions in quiescent 317 conditions or homogeneous flow, in this study, we report on the effect of the shear flow 318 condition, a kind of simplest inhomogeneous flow field, on the hydrodynamic drag 319 320 force, on the particle within the vicinity of the wall. These pieces of information can be

utilized to optimized microfluidic designs for mixing and separations of particles or
exploit the formation of the concentration gradient of particles perpendicular to flow
directions caused by the non-linear hydrodynamic interactions. Finally, the present
results may be utilized as an alternative method for force calibration for the optical trap.

## 325 **4. Conclusion**

Combining optical trapping equipped with force detection and a microfluidic 326 327 channel is utilized to quantitatively evaluate the hydrodynamic drag force on a particle near the wall in a quiescent and shear flow environment. The calculated drag friction 328 coefficient increases with decreasing distance from the wall due to the increased 329 hydrodynamic effect. For particle-wall interaction without shear flow, the change in the 330 drag coefficient is well represented by the theoretical prediction and coincides with the 331 previous studies. In the case of particle-wall interaction under shear flow, the 332 experimentally calculated drag friction coefficient is in quantitative agreement with the 333 exact solution of the theoretical model of Goldman et al. (1967b). These findings 334 provide a straightforward force measurement technique for verifying the hydrodynamic 335 drag force coefficient, which is further available to calibrate optical trapping force 336 337 under shear. Furthermore, with the advent of lab-on-a-chip devices, this result will provide helpful information in quantifying the effect of the wall and controlling the 338 particle in the microfluidic flow. This can be used to optimized microfluidic designs for 339 340 mixing and separations of particles or exploit the formation of the concentration gradient of particles perpendicular to flow directions caused by the non-linear 341

342 hydrodynamic interactions for high throughput operations.

# 343 Author Contributions

Lester C. Geonzon: Conceptualization; Data curation; Formal analysis;
Methodology; Software; Validation; Visualization; Writing-original draft; Writingreview & editing. Motoyoshi Kobayashi: Supervision; Conceptualization; Formal
analysis; Visualization; Writing-original draft; Writing-review & editing; Resources;
Funding acquisition; Project administration. Yasuhisa Adachi: Supervision,
Resources; Funding acquisition; Project administration.

## 350 **Conflicts of interest**

351 There are no conflicts to declare.

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## 355 **References**

356	1.	R. Beckett, G. Nicholson, B. T. Hart, M. Hansen and J. C. Giddings, Water
357		Research, 1988, 22, 1535-1545.
358	2.	G. Karaiskakis, K. A. Graff, K. D. Caldwell and J. C. Giddings, International
359		Journal of Environmental Analytical Chemistry, 1982, 12, 1-15.
360	3.	H. M. Wyss, D. J. Blair, J. F. Morris, H. A. Stone and D. W. Weitz, Phys. Rev.
361		<i>E</i> , 2006, 74, 061402.
362	4.	A. A. S. Bhagat, H. W. Hou, L. D. Li, C. T. Lim and J. Han, Lab Chip, 2011,
363		11, 1870-1878.

364	5.	S. N. Bhatia and D. E. Ingber, Nature Biotechnology, 2014, 32, 760-772.
365	6.	K. E. Nelson and T.R. Ginn, Langmuir, 2005, 21, 2173-2184.
366	7.	Z. A. Kuznar, and M. Elimelech, Colloids and Surf. A, 2007, 294, 156-162.
367	8.	M. Kobayashi, H. Nanaumi and Y. Muto, Colloids and Surf. A, 2009, 347, 2-
368		7.
369	9.	L. C. Geonzon, R. G. Bacabac and S. Matsukawa, Journal of Electrochemical
370		Society, 2019, 166, B3228.
371	10.	L. C. Geonzon, A. Santoya, X. Zhuang, J. Xie, R. G. Bacabac and S.
372		Matsukawa, Food Hydrocolloids, 2020, 105, 105759.
373	11.	S. Huang and D. E. Ingber, Nat Cell Biol. 1999, 1:E131-8.
374	12.	A. A. Khalili and M. R. Ahmad, Int J Mol Sci, 2015, 16, 18149-18184.
375	13.	Y. S. Choi and S. J. Lee, Microfluid Nanofluid, 2010, 9, 819-829.
376	14.	L. Clime, K. J. Morton, X. D. Hoa and T. Veres, Scientific Reports, 2015, 5,
377		9765.
378	15.	W. R. Dean and M.E. O'Neill, Mathematika, 1963, 10, 13.
379	16.	M. E. O'Neill, Mathematika, 1964, 11, 67.
380	17.	A.J. Goldman, R. G. Cox and H. Brenner, Chem. Eng. Sci., 1967a, 22, 637-
381		651.
382	18.	A.J. Goldman, R. G. Cox and H. Brenner, Chem. Eng. Sci., 1967b, 22, 653-
383		660.
384	19.	M. E. O'Neill and K. Stewartson, J. Fluid Mech., 1967, 27, 705-724.
385	20.	M. E. O'Neill, Chem. Eng. Sci., 1968, 23, 1293-1298.
386	21.	G. S. Perkins and R. B. Jones, <i>Physica A</i> , 1992, <b>189</b> , 447-477.
387	22.	J. Magnaudet, S. H. U. Takagi and D. Legendre, J. Fluid Mech., 2003, 476,
388		115-154.
389	23.	M. Chaoui and F. Feuillebois, Q. J Mech. App. Math, 2003, 56, 381-410.
390	24.	R. Kurose and S. Komori, J. Fluid Mech., 1999, 384, 183-206. DOI:
391	25.	L. Zeng, F. Najjar, S. Balachandar and P. Fischer, Phys. Fluid, 2009, 21,
392		033302.
393	26.	F. Takemura and J. Magnaudet, Phys. Fluid, 2009, 21, 083303.
394	27.	H. Lee and S. Balachandar, J. Fluid Mech., 2010, 657, 89-125.
395	28.	O.B. Bocharov and D.Y. Kushnir, <i>Thermophysics and Aeromechanics</i> , 2016, 23,
396		83-95.

397	29.	Z. Zhou, G. Jin, B. Tian and J. Ren, Int. Journal of Multiphase Flow, 2017,
398		<b>92</b> , 1-19.
399	30.	N. I. K. Ekanayake, J. D. Berry, A. D. Stickland, D. E. Dunstan, I. L. Muir, S.
400		K. Dower and D. J. E. Harvie, J. Fluid Mech., 2020, 904, A6.
401	31.	P. Huang, J. Guasto and K Breuer, J. Fluid Mech., 2006, 566, 447, 464.
402	32.	P. Huang and K. Breuer, Phys. Rev. E, 2007, 76, 043607.
403	33.	S. Adarwa, H. B. Tabrizi and G. Ahmadi, Particuology, 2014, 16, 84-90.
404	34.	A. Ashkin and J. M. Dziedzic, Science, 1987, 235, 1517-1520.
405	35.	K. Svoboda and S. M. Block, Annu. Rev. Biophys. Biomol. Struct., 1994, 23,
406		247-85.
407	36.	E. Fällman, S. Schedin, J. Jass, M. Andersson, B. E. Uhlin and O. Axner,
408		Biosens. Bioel., 2004, 19, 1429-1437.
409	37.	E.Schäffer, S. F. Nørrelykke and J. Howard, <i>Langmuir</i> , 2007, 23, 3654–3665.
410	38.	D. Mizuno, R. Bacabac, C. Tardin, D. Head and C. F. Schmidt, PRL, 2009,
411		<b>102</b> , 168102.
412	39.	M. Tassieri, G. M. Gibson, R. M. L. Evans, A. M. Yao, R. Warren, M. J.
413		Padgett and J. M. Cooper, Phys. Rev. E, 2010, 87, 026308.
414	40.	M. Tassieri, R. M. L. Evans, R. Warren, N. J. Bailey and J. M. Cooper, New
415		Journal of Physics, 2012, 14, 115032.
416	41.	M. Kobayashi, M. Skarba, P. Galleto, D. Cakara and M. Borkovec, Journal of
417		Colloid and Interface Science, 2005, 292, 139-147.
418	42.	S. H. Behrens and D. G. Grier, J. Chem. Phys. 2001, 115, 6716.
419	43.	M. J. Lang, C. L. Asbury, J. W. Shaevitz and S. M. Block, Biophys. J., 2002,
420		<b>83</b> , 491-501.
421	44.	K. C. Vermeulen, G. J. L. Wuite, G. J. M. Stienen and C. F. Schmidt, Appl. Optics,
422		2006, <b>45</b> , 1812-1819.
423	45.	N. Eom, V. Stevens, A. B. Wedding, R. Sedev and J. N. Connor, Advanced
424		Powder Technology, 2014, 25, 1249-1253.
425	46.	K. Berg-Sørensen and H. Flybjerg, Rev. Sci. Instrum., 2004, 75, 594-612.
426	47.	A. Sheikhi and R. Hill, Soft Matter, 2016, 12, 6575-6587.
427	48.	H. Bruus, Theoretical Microfluidics, 1st Ed., Oxford University Press, 2008.
428	49.	P. P. Lele, J. W. Swan, J. F. Brady, N. J. Wagner and E. M. Furst, Soft Matter,
429		2011, 7, 6844.

430	50.	J. W. Swan and J. F. Brady, <i>Phys. Fluids</i> , 2007, <b>19</b> 113306.
431	51.	M. Tassieri, F. Del Giudice, E. K. Robertson, N. Jain, B. Fries, R. Wilson, A.
432		Glidle, F. Greco, P. A. Netti, P. L. Maffettone, T. Bicanica and J. M. Cooper,
433		Scientific Reports, 2015, 5:8831.
434	52.	M. Tanyeri, M. Ranka, N. Sittipolkul and C. M. Schroeder, Lab Chip, 2011, 11,
435		1786.
436	53.	X. Lu, C. Liu, G. Hu and X. Xuan, Journal of Colloid and Interface Science,

437 2015, **500**, 182-201



Figure 1. Trap stiffness, k of the optical trap as a function of distance from the wall h/a.



Figure 2. a) Time series of lateral position of an optically trapped particle at different height from the wall without shear flow. b) Height dependence of the normalized drag coefficient  $\gamma$  (black) and fit parameter  $\lambda$  of the normalized position autocorrelation function (blue). The dashed line indicates the fitting line from the Faxen's Law while the solid line is the exact solution of friction factor for translation by Goldman *et al*.<sup>17.</sup>



Figure 3. Theoretical fluid velocity profiles across the microchannel for different volume flow rates Q. Open symbols are the experimental local fluid velocity measured at volumetric flow rate 500  $\mu$ L/hr. (Inset) Experimental region of interest.



Figure 4. Displacement of the trapped particle from the equilibrium position with shear flow as a function of h/a at various volume flow rates Q.



Figure 5. Experimentally calculated drag correction factor at different h/a. The solid red line is obtained from the exact solution of the theoretical model by Goldman *et al.*<sup>18</sup>.